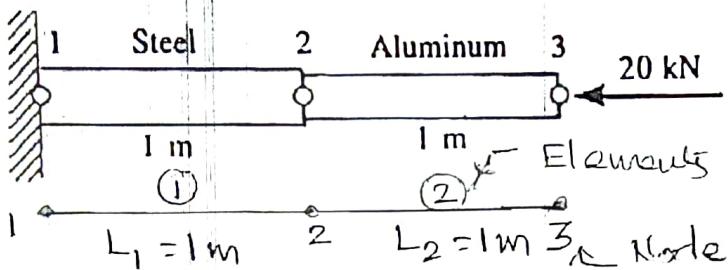




For the bar assemblage shown in Figure 0.0, determine the nodal displacements, the forces in each element, and the reactions. Use the direct stiffness method for these problems.



$$E_{st} = 200 \text{ GPa}$$

$$A_{st} = 4 \times 10^{-4} \text{ m}^2$$

$$E_{al} = 70 \text{ GPa}$$

$$A_{al} = 2 \times 10^{-4} \text{ m}^2$$

Stiffness matrix
For Element ①

$$K^{(1)} = \frac{A_1 E_1}{L_1} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$K^{(1)} = 4 \times 10^4 \left(\frac{\text{m}^2}{\text{m}^2}\right) \times 200 \times 10^9 \left(\frac{\text{kN}}{\text{m}^2}\right) \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 800 \times 10^2 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \frac{\text{kN}}{\text{m}}$$

$$= 10^2 \begin{bmatrix} 800 & -800 \\ -800 & 800 \end{bmatrix} \frac{\text{kN}}{\text{m}}$$

Global matrix

$$K^{(G)} = 10^2 \begin{bmatrix} 1 & 2 & 3 \\ 2 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\begin{bmatrix} 800 & -800 & 0 \\ -800 & 800 & -140 \\ 0 & -140 & 140 \end{bmatrix}$$

Stiffness matrix
For Element ②

$$K^{(2)} = \frac{A_2 E_2}{L_2} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 2 \times 10^4 \times 70 \times 10^9 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 10^2 \begin{bmatrix} 140 & -140 \\ -140 & 140 \end{bmatrix} \frac{\text{kN}}{\text{m}}$$

$$= \begin{bmatrix} 800 & -800 & 0 \\ -800 & 840 & -140 \\ 0 & -140 & 140 \end{bmatrix}$$

4/4



Element Equation : $\{K\}\{U\} = \{F\}$

Global stiff matrix

Displacement Vector Force vector

$$U = \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} \quad F = \begin{Bmatrix} F_1 \\ F_2 \\ F_3 \end{Bmatrix}$$

Formulation of Element Equation.
 $\{K\}\{U\} = \{F\}$

$$10^2 \begin{bmatrix} 800 & -800 & 0 \\ -800 & 940 & -140 \\ 0 & -140 & 140 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} f_1 \\ f_2 \\ f_3 \end{Bmatrix}$$

Apply Boundary Condition. $u_1 = 0, f_1 = 0$

$$f_2 = 0, f_3 = -20$$

$$10^2 \begin{bmatrix} 800 & -800 & 0 \\ -800 & 940 & -140 \\ 0 & -140 & 140 \end{bmatrix} \begin{Bmatrix} 0 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ -20 \text{ kN} \end{Bmatrix}$$

Apply Elimination Approach

$$10^2 \begin{bmatrix} 940 & -140 \\ -140 & 140 \end{bmatrix} \begin{Bmatrix} u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ -20 \text{ kN} \end{Bmatrix}$$



Generalization of simultaneous equation from Eqn

$$10^2 [940 u_2 - 140 u_3] = 0 \dots \textcircled{1}$$

$$u_3 = 6.741 u_2$$

$$10^2 [-140 u_2 + 140 u_3] = -20 \text{ (crossed out)} \dots \textcircled{2}$$

Substituting $u_3 = 6.741 u_2 \rightarrow \textcircled{2}$

$$10^2 [-140 u_2 + 140 [6.741 u_2]] = -20 \text{ (crossed out)}$$

$$u_2 = -0.25 \times 10^{-3} \text{ m}$$

$$u_3 = -1.678 \times 10^{-3} \text{ m}$$

Strain Calculation

$$\epsilon^{(1)} = \frac{u_2 - u_1}{L_1} = \frac{-0.25 \times 10^{-3}}{1} = -0.25 \times 10^{-3}$$

$$\epsilon^{(2)} = \frac{u_3 - u_2}{L_2} = \frac{-1.678571429 \times 10^{-3} - 0.25 \times 10^{-3}}{1} = -1.428571429 \times 10^{-3}$$

Stress Calculations [By Hooke's law]

$$\sigma^{(1)} = E_1 \epsilon^{(1)} = 200 \times 10^6 \times -0.25 \times 10^{-3} = 50000$$

$$\sigma^{(2)} = E_2 \epsilon^{(2)} = 70 \times 10^6 \times -1.428571429 \times 10^{-3} = 10,000$$

Theoretical stresses for this problem are easily calculated by

(1)

$$\text{Theory} = \frac{P}{A_1} = \frac{20}{4 \times 10^{-4}} = 50,000$$

(2)

$$\text{Theory} = \frac{P}{A_2} = \frac{20}{2 \times 10^{-4}} = 10,000$$

Reaction force

$$[R] = [K][U] - [F]$$

$$\begin{bmatrix} R_1 \\ R_2 \\ R_3 \end{bmatrix} = 10^2 \begin{bmatrix} 800 & -800 & 0 \\ -800 & 940 & -140 \\ 0 & -140 & 140 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \end{bmatrix} - \begin{bmatrix} f_1 \\ f_2 \\ f_3 \end{bmatrix}$$

Apply BC

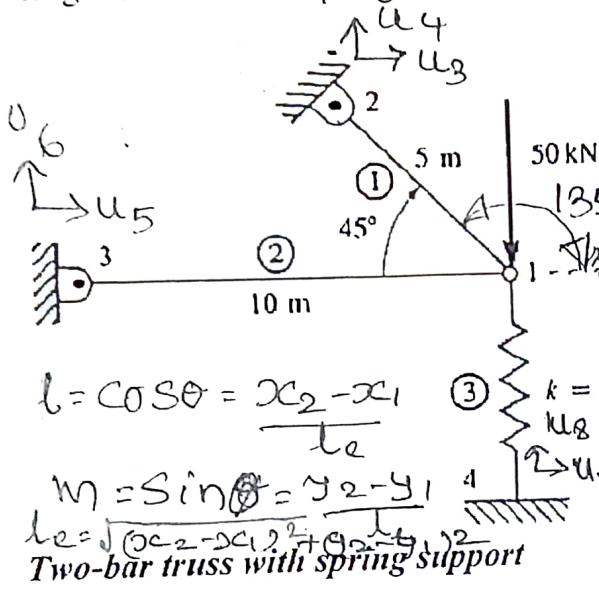
$$\begin{bmatrix} R_1 \\ 0 \\ 0 \end{bmatrix} = \frac{1}{10^2} \begin{bmatrix} 800 & -800 & 0 \\ -800 & 940 & -140 \\ 0 & -140 & 140 \end{bmatrix} \begin{bmatrix} 0 \\ u_2 \\ u_3 \end{bmatrix} - \begin{bmatrix} 0 \\ 0 \\ -20 \end{bmatrix}$$

$$- 800 \times 10^2 \times u_2 - 0 = R_1$$

$$- 800 \times 10^2 \times 1.678571429 \times 10^{-3} = \underline{\underline{20 \text{ KN}}}$$



To illustrate how we can combine spring and bar elements in one structure, we now solve the two-bar truss supported by a spring shown in Figure 1. Both bars have $E = 210 \text{ GPa}$ and $A = 5 \times 10^4 \text{ m}^2$. Bar one has a length of 5 m and bar two a length of 10 m. The spring stiffness is $k = 2000 \text{ kN/m}$.



Solution:

Stiffness matrix

$$K = \frac{AE}{L} \begin{bmatrix} l^2 & lm - l^2 & -lm \\ lm & m^2 - lm - w^2 & -l^2 \\ -l^2 & -lm & l^2 - lm \\ -lm & -m^2 & lm - w^2 \end{bmatrix}$$

$$l = \cos \theta = \frac{x_2 - x_1}{r_e}$$

$$m = \sin \theta = \frac{y_2 - y_1}{r_e}$$

$$r_e = \sqrt{(x_2 - x_1)^2 + (y_2 - y_1)^2}$$

Two-bar truss with spring support

Element ① $\theta = 135^\circ$

$$l = \cos 135^\circ = -0.707 \cdot 10^{-6} \times 10^3$$

$$l^2 = 0.5$$

$$m = \sin 135^\circ = +0.707 \cdot 10^{-6} \times 10^3$$

$$m^2 = 0.5 \quad [lm = -0.5]$$

Stiffness matrix

for Element ① [node(1-2)]

$$L = 5 \text{ m}$$

$$K^{(1)} = (5 \times 10^{-4}) \times (210 \times 10^9)$$

$$\frac{5}{105 \times 10^5}$$

$$= 105 \times 10^5$$

$$\begin{bmatrix} 0.5 & -0.5 & -0.5 & 0.5 \\ -0.5 & 0.5 & 0.5 & -0.5 \\ -0.5 & 0.5 & 0.5 & -0.5 \\ 0.5 & -0.5 & -0.5 & 0.5 \end{bmatrix}$$

$$\begin{bmatrix} 1 & 2 & 3 & 4 \\ 1 & -1 & -1 & 1 \\ -1 & 1 & 1 & -1 \\ -1 & 1 & 1 & -1 \\ 1 & -1 & -1 & 1 \end{bmatrix} \begin{bmatrix} 1 \\ 2 \\ 3 \\ 4 \end{bmatrix}$$



Element (2), $\theta = 180^\circ$

$$l^2 = 1 \quad l_m = 0 \quad m^2 = 0$$

Stiffness matrix for Element (2),
Nodes [1-2]

$$L = 10m$$

$$K^{(2)} = \frac{(5 \times 10^{-4}) \times (210 \times 10^9)}{10}$$

$$= 105 \times 10^5 \begin{bmatrix} 1 & 2 & 5 & 6 \\ 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

Element (3), $\theta = 270^\circ$

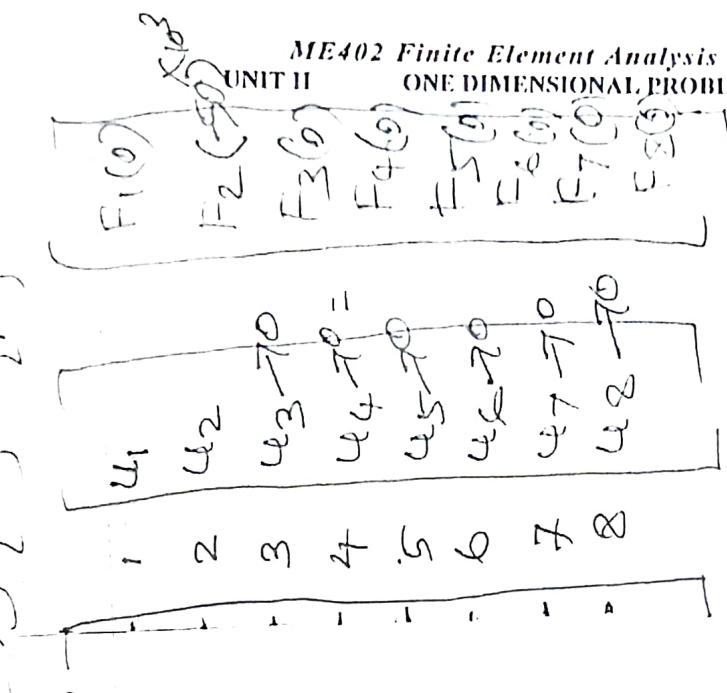
$$l^2 = 0, \quad l_m = 0, \quad m^2 = 1$$

Stiffness matrix for Element (3)
Nodes [1-4] = $K^{(3)}$ []

$$K^{(3)} \approx 20 \times 10^5 \begin{bmatrix} 1 & 2 & 7 & 8 \\ 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix}$$



One dimensional Finite Element Equations: $\{K\} \{u\} = \{F\}$



$$\begin{bmatrix} 1 & 2 & 3 & 4 & 5 & 6 & 7 & 8 \\ 105 & -105 & 105 & 105 & -105 & 0 & 0 & -20 \\ -105 & 105 & -105 & 0 & 0 & 0 & 0 & 0 \\ 105 & -105 & 105 & 0 & 0 & 0 & 0 & 0 \\ 105 & -105 & 105 & 0 & 0 & 0 & 0 & 0 \\ 105 & -105 & 105 & 0 & 0 & 0 & 0 & 0 \\ 105 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & -20 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \\ u_8 \end{bmatrix} = \begin{bmatrix} F_1 \\ F_2 \\ F_3 \\ F_4 \\ F_5 \\ F_6 \\ F_7 \\ F_8 \end{bmatrix}$$

Boundary conditions are

$$\begin{aligned} u_3 = u_4 = u_5 = u_6 = u_7 = u_8 &= 0 \\ \text{The final matrix} &= \begin{bmatrix} u_1 \\ u_2 \\ -105 \\ 105 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ -50 \\ 125 \end{bmatrix} \end{aligned}$$

Stress : Stress of element (1)

$$\sigma_{(1)} = \frac{E}{L_1} [-1, -m_1, \lambda_1, m_1] \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \end{bmatrix}$$

$$\sigma_{(1)} = \frac{210 \times 10^9}{5} [0.707 \quad -0.707 \quad 0.707 \quad 0.707] \begin{bmatrix} -3.448 \times 10^{-3} \\ -6.896 \times 10^{-3} \\ 0 \\ 0 \end{bmatrix}$$

$$= 102.4 \text{ MPa} [T]$$

$$\sigma_{(2)} = \frac{E}{L_2} [-\lambda_2, -m_2, \lambda_2, m_2] \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \end{bmatrix}$$

$$= \frac{210 \times 10^9}{10} [1 \quad 0 \quad -1 \quad 0] \begin{bmatrix} -3.448 \times 10^{-3} \\ -6.896 \times 10^{-3} \\ 0 \\ 0 \end{bmatrix}$$

$$= -72. \text{ MPa} [C]$$

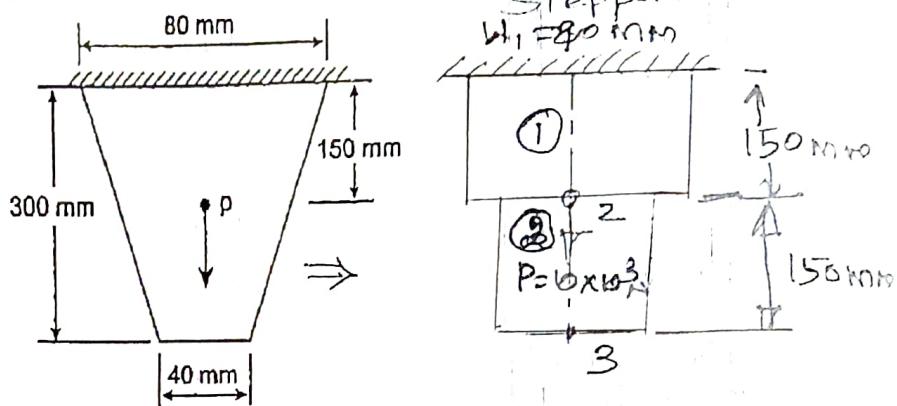
Note : Can show equilibrium at node 1

$$F_S = [2000 \text{ kN/m}] \times [6.896 \times 10^{-3} \text{ m}] = 13.792 \text{ kN}$$

$$\begin{aligned} f_{1-3} &= 35.6 \text{ kN} \\ f_{1-2} &= 51.2 \text{ kN} \end{aligned} \quad \begin{aligned} \leq F_y &= 0 \\ -50 + 13.792 + 36.198 &= 0 \end{aligned}$$



For a tapered plate of uniform thickness $t=10\text{mm}$ as shown in Figure 1, find the displacements at the nodes by meshing into two element model. The bar has mass density $\rho = 7800\text{kg/m}^3$, $E = 2 \times 10^5 \text{MN/m}^2$. In addition to self-weight, the plate is subjected to a point load $P=10\text{kN}$ at its center. Also determine the reaction force at the support.



Solution

Figure 1

Area at node 1, A_1

$$= \text{width} \times \text{thickness} = w_1 \times t, \\ = 80 \times 10 \\ A_1 = 800 \text{ mm}^2$$

Area at node 2, A_2

$$A_2 = \left[\frac{w_1 + w_3}{2} \right] \times t_2 \\ = \left[\frac{80 + 40}{2} \right] \times 10 \\ A_2 = 600 \text{ mm}^2$$

Area at node 3, A_3

$$\text{Area at node } 3, A_3 \\ A_3 = w_3 \times t_3 = 40 \times 10 \\ = 400 \text{ mm}^2$$

Average area of elements (1) $A_{1,2}^{(1)}$

$$= \text{Area at node 1} + \text{Area at node 2}$$

$$= \frac{800 + 600}{2}$$

$$A_{1,2}^{(1)} = 700 \text{ mm}^2$$

Average Area of element (2) $A_2^{(2)}$

$$= \frac{A_2 + A_3}{2}$$

$$= \frac{600 + 400}{2}$$

$$A_2^{(2)} = 500 \text{ mm}^2$$



$$\text{Mass density } \rho = 7800 \text{ kg/m}^3 = 7800 \times 9.81 \text{ N/m}^3 \\ = 76518 \text{ N/m}^3 = 76518 \times 10^9 \text{ N/mm}^3 \\ = 7.6518 \times 10^5 \text{ N/mm}^3$$

$$\text{Young's Modulus } E = 2 \times 10^5 \text{ MN/m}^2 \\ = 2 \times 10^5 \times 10^6 \text{ N/m}^2 \\ = 2 \times 10^5 \times 10^6 \times 10^{-6} \text{ N/mm}^2 \\ = 2 \times 10^5 \text{ N/mm}^2$$

Stiffness matrix for element ①

$$K_1 = \frac{A_1^{(1)} E}{L_1} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= \frac{700 \times 2 \times 10^5}{150} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 4.666 \times 2 \times 10^5 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 105 \begin{bmatrix} 9.332 & -9.332 \\ -9.332 & 9.332 \end{bmatrix}$$

Stiffness matrix for element ②

$$K_2 = \frac{A_2^{(2)} E}{L_2} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= \frac{500 \times 2 \times 10^5}{150} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 6.666 \times 10^5 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 105 \begin{bmatrix} 6.666 & -6.666 \\ -6.666 & 6.666 \end{bmatrix}$$

$$\text{Global matrix } [K] = K_1 + K_2$$

$$= 105 \begin{bmatrix} 9.332 & -9.332 & 0 \\ -9.332 & 9.332 + 6.666 & -6.666 \\ 0 & -6.666 & 6.666 \end{bmatrix}$$

$$\text{Displacement vector } U = \begin{bmatrix} u_1 \\ u_2 \\ u_3 \end{bmatrix}$$

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Body Force Vector $\{F\} = \frac{\rho A L}{2} [1]$

Force Vector

For element 1

$$\begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \frac{\rho_1 A_1^{(1)} L_1}{2} [1]$$

$$= 7.6518 \times 10^{-5} \times 700 \times 150 [1]$$

$$= \begin{Bmatrix} 4.017 \\ 4.017 \end{Bmatrix}$$

Force vector

for element 2

$$\begin{Bmatrix} F_2 \\ F_3 \end{Bmatrix} = \frac{\rho_2 A_2^{(2)} L_2}{2} [1]$$

$$= 7.6518 \times 10^{-5} \times 500 \times 150 [1]$$

$$= \begin{Bmatrix} 2.869 \\ 2.869 \end{Bmatrix}$$

Total force vector

$$\begin{Bmatrix} F_1 \\ F_2 \\ F_3 \end{Bmatrix} = \begin{Bmatrix} 4.017 \\ 4.017 + 2.869 \\ 2.869 \end{Bmatrix} = \begin{Bmatrix} 4.017 \\ 6.886 \\ 2.869 \end{Bmatrix}$$

Assemble the finite element equation $[K] \{U\} = \{F\}$

$$10^5 \begin{bmatrix} 9.332 & -9.332 & 0 \\ -9.332 & 15.998 & -6.666 \\ 0 & -6.666 & 6.666 \end{bmatrix} \begin{Bmatrix} U_1 \\ U_2 \\ U_3 \end{Bmatrix} = \begin{Bmatrix} 4.017 \\ 6.886 \\ 2.869 \end{Bmatrix}$$

Apply the boundary condition i.e at node 1
displace $u_1 = 0$.

A point load of $10 \times 10^3 N$ is acting
at node 2, so add $10,000 N$ in F_2
Vector.

$$105 \begin{bmatrix} 9.332 & -9.332 & 0 \\ -9.332 & 15.998 & -6.666 \\ 0 & -6.666 & 6.666 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \end{bmatrix} = \begin{bmatrix} 4.017 \\ 10006.886 \\ 2.869 \end{bmatrix}$$

In the above equation $u_1 = 0$, so neglect
first row and first column of [KJ]
matrix. The reduced equation is.

$$105 \begin{bmatrix} 15.998 & -6.666 \\ -6.666 & 6.666 \end{bmatrix} \begin{bmatrix} u_2 \\ u_3 \end{bmatrix} = \begin{bmatrix} 10006.886 \\ 2.869 \end{bmatrix}$$

$$(15.998 u_2 - 6.666 u_3) 10^5 = 10006.886 \quad \textcircled{1}$$

$$(-6.666 u_2 + 6.666 u_3) 10^5 = 2.869 \quad \textcircled{2}$$

Solve above equations

$$u_2 = 0.01073 \text{ mm}$$

$$u_3 = 0.01073 \text{ mm}$$

Reaction force $\{R\} = [K]\{U\} - \{F\}$

$$= 10^5 \begin{bmatrix} 9.332 & -9.332 & 0 \\ -9.332 & 15.998 & -6.666 \\ 0 & -6.666 & 6.666 \end{bmatrix} \begin{bmatrix} 0 \\ 0.01073 \\ 0.01073 \end{bmatrix} - \begin{bmatrix} 4.017 \\ 10,006.886 \\ 2.869 \end{bmatrix}$$

$$= \cancel{10^5} \begin{bmatrix} -10,000 \\ 10,000 \\ 0 \end{bmatrix} - \begin{bmatrix} 4.017 \\ 10,006.886 \\ 2.869 \end{bmatrix}$$

$$\therefore \begin{bmatrix} R_1 \\ R_2 \\ R_3 \end{bmatrix} = \begin{bmatrix} -10004.017 \\ -6.886 \\ -2.869 \end{bmatrix} N$$

Reaction force is equivalent and opposite to applied force.

Verification $R_1 + R_2 + R_3$

$$= -10004.017 - 6.886 - 2.869$$

$$= -10013.772 N$$

$$\text{Applied force} = 4.017 + 10,006.886 + 2.869 \\ = 10013.772 N$$

Result : Displacement $U_1 = 0$

$$U_2, U_3 = 0.01073 \text{ mm}$$

Reaction force at the support



A compound axial member is subjected to the loads shown in Fig. 1.1. Given, $E_1 = 50 \text{ MN/m}^2$, $E_2 = 100 \text{ MN/m}^2$, $L_1 = 0.5 \text{ m}$, $L_2 = 1 \text{ m}$, $A_1 = 20 \text{ cm}^2$, $A_2 = 10 \text{ cm}^2$, find (i) reaction force at points 1 (F1) and (ii) displacements at nodes 2 and 3 (u_2 and u_3) using two bar elements model.

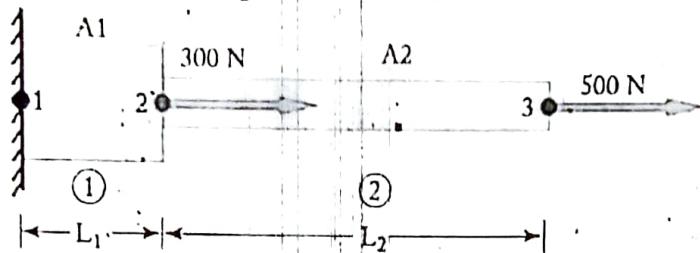
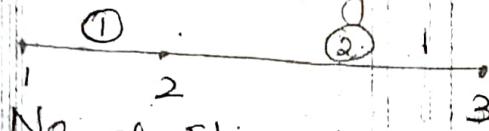


Fig 1.1 A compound axial member subjected to loads.

Solution.

Process - Processing FEA model



No of Elements = 2
Number of nodes = 3

The Stiffness matrix of each element is computed from

$$[K]^e = \frac{4E}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

For element (1), the stiffness matrix is

$$\begin{aligned} [K]^{(1)} &= 0.002 \times 50 \times 10^6 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \\ &= 2 \times 10^5 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \\ &= 10^5 \begin{bmatrix} 2 & -2 \\ -2 & 2 \end{bmatrix} \end{aligned}$$

For element (2), the stiffness matrix is

$$\begin{aligned} [K]^{(2)} &= 0.001 \times 100 \times 10^6 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \\ &= 1 \times 10^5 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \\ &= 10^5 \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \end{aligned}$$

The final global matrix of stiffness matrix

$$[K]^{(G)} = \begin{bmatrix} 1 & 2 & 3 \\ 2 & -2 & 0 \\ -2 & 2+1 & -1 \end{bmatrix} \begin{bmatrix} 10^5 \\ 1 \\ 2 \\ 3 \end{bmatrix}$$



$$\text{Displacement } \{u\} = \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix}$$

$$\text{Load/Force matrix } \{F\} = \begin{Bmatrix} F_1 \\ F_2 \\ F_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 300 \\ 500 \end{Bmatrix}$$

$$\text{Finite Element Equation } [K] \{u\} = \{F\}$$

$$10^5 \begin{bmatrix} 2 & -2 & 0 \\ -2 & 3 & -1 \\ 0 & -1 & 0 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 300 \\ 500 \end{Bmatrix}$$

Apply the fixed boundary condition at node 1 and applying the external forces at nodes 2 and 3,

$$10^5 \begin{bmatrix} 2 & -2 & 0 \\ -2 & 3 & -1 \\ 0 & -1 & 0 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 300 \\ 500 \end{Bmatrix}$$

using the matrix partitioning to solve for u_2 and u_3 , we have

$$10^5 \begin{bmatrix} 3 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 300 \\ 500 \end{Bmatrix}$$

$$\begin{Bmatrix} u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0.004 \\ 0.009 \end{Bmatrix} \text{ m}$$

Post-processing: using the first row of the global matrix to find R_1 , we have

$$R_1 = -2 \times 10^5 \times 0.004$$

Prepared by Dr. M. SUBRAMANIAN/Professor/Mechanical/ME402/ Finite Element Analysis = -800N

check for Equilibrium

$$\text{Action Force} = -\text{Reaction force} \cdot R_1 = -800N$$