



Velocity triangles

It is the component which is responsible for “actual work done” across blades.

The jet of fluid that strikes the turbine blades, has its two components:

1. Whirl component
2. Axial thrust

The whirl velocity is the tangential component of absolute velocity at the blade inlet and outlet. This component of velocity is responsible for the whirling or rotating of the turbine rotor.

Otherwise

In turbomachinery, a velocity triangle or a velocity diagram is a triangle representing the various components of velocities of the working fluid in a turbomachine.

Velocity triangles may be drawn for both the inlet and outlet sections of any turbomachine.

The vector nature of velocity is utilized in the triangles, and the most basic form of a velocity triangle consists of the tangential velocity, the absolute velocity and the relative velocity of the fluid making up three sides of the triangle.

A general velocity triangle consists of the following vectors:

V : Absolute Velocity of The Fluid.

U : Blade Linear Velocity.

V_r : Relative Velocity of The Fluid After Contact With Rotor.

V_w : Tangential Component of V (Absolute Velocity), Called Whirl Velocity.

V_f : Flow Velocity

(Axial Component In Case of Axial Machines, Radial Component In Case of Radial Machines).

The Following Angles Are Encountered During The Analysis:

A : Angle Made by V With The Plane of The Machine (Usually The Nozzle Angle or The Guide Blade Angle).

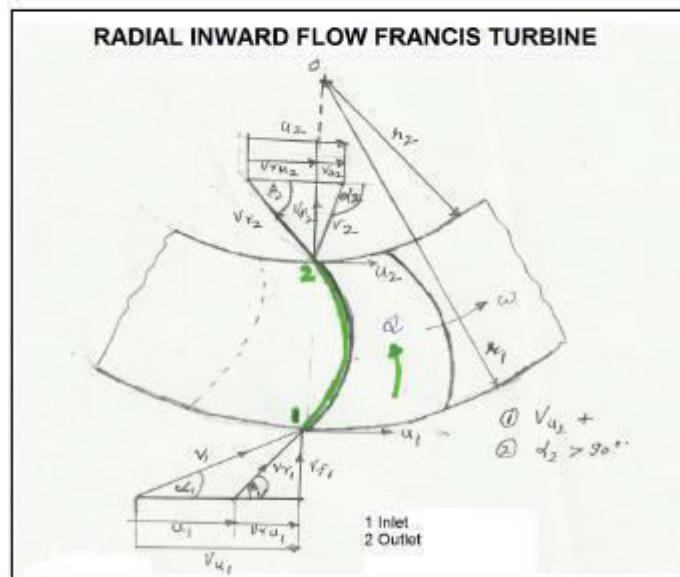
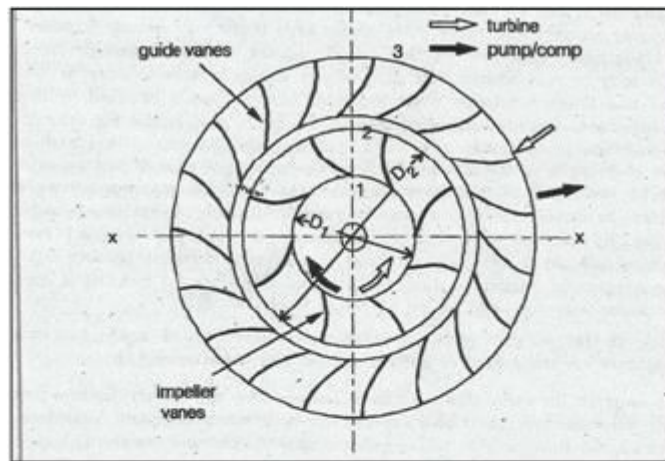
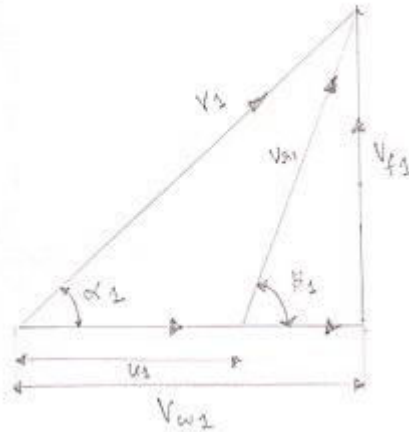
B : Angle of The Rotor Blade. Absolute Angle



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V_{u2} = Whirl component of absolute velocity at outlet;

V_{f2} = Flow component of absolute velocity at outlet ; Inlet and Outlet Velocity Triangles

Referring to velocity triangles

1 – Inlet, 2 – outlet

V_1 = Absolute velocity of the fluid at inlet (before entering the rotor vanes)

V_{r1} = Relative velocity of the fluid at rotor inlet

V_{u1} = Tangential component of absolute velocity

OR

Whirl component of velocity at inlet

V_{ru2} = Whirl component of relative velocity at outlet

U_2 = Linear rotor velocity at outlet

β_1 = Fluid or jet angle at outlet (To the direction of wheel rotation)

β_1 = Vane (blade) angle at outlet (To the direction of wheel rotation)

V_{f1} = Flow component of absolute velocity at inlet

V_{ru1} = Whirl component of relative velocity at inlet

U_1 = Linear rotor vane velocity at inlet

β_1 = Absolute jet angle at inlet

β_1 = Vane (blade) angle at inlet

Referring to outlet velocity triangle



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The above figure shows the velocity triangles at entry and exit of a general TM. The angular velocity of the rotor is ω rad/sec and is given by

$$\omega = \frac{2\pi N}{60} \quad \text{--- 1}$$

The peripheral velocity of the blade at the entry and exit corresponding to the diameter D_1 and D_2 are given by

$$u_1 = \frac{\pi D_1 N}{60} \quad \text{--- 2}$$

$$u_2 = \frac{\pi D_2 N}{60} \quad \text{--- 3}$$

The three velocity vector V , u and V_r at the section are related by a simple vector equation

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$$V = u + V_r \text{ --- 4}$$

Considering unit mass of the fluid entering and leaving in unit time we have

$$\text{Angular momentum of the inlet} = V_{u1} \times R_1 \text{ --- 5}$$

Similarly

$$\text{Angular momentum of the outlet} = V_{u2} \times R_2 \text{ --- 6}$$

Torque produced = rate of change of Angular momentum

$$T = \{\text{Angular momentum at inlet}\} - \{\text{Angular momentum at outlet}\}$$

$$T = (V_{u1}R_1) - (V_{u2}R_2) \text{ --- 7}$$

Therefore the work done is given by

$$W = \text{Torque} \times \text{angular velocity of the rotor}$$

$$W = (V_{u1}R_1 - V_{u2}R_2)\omega$$

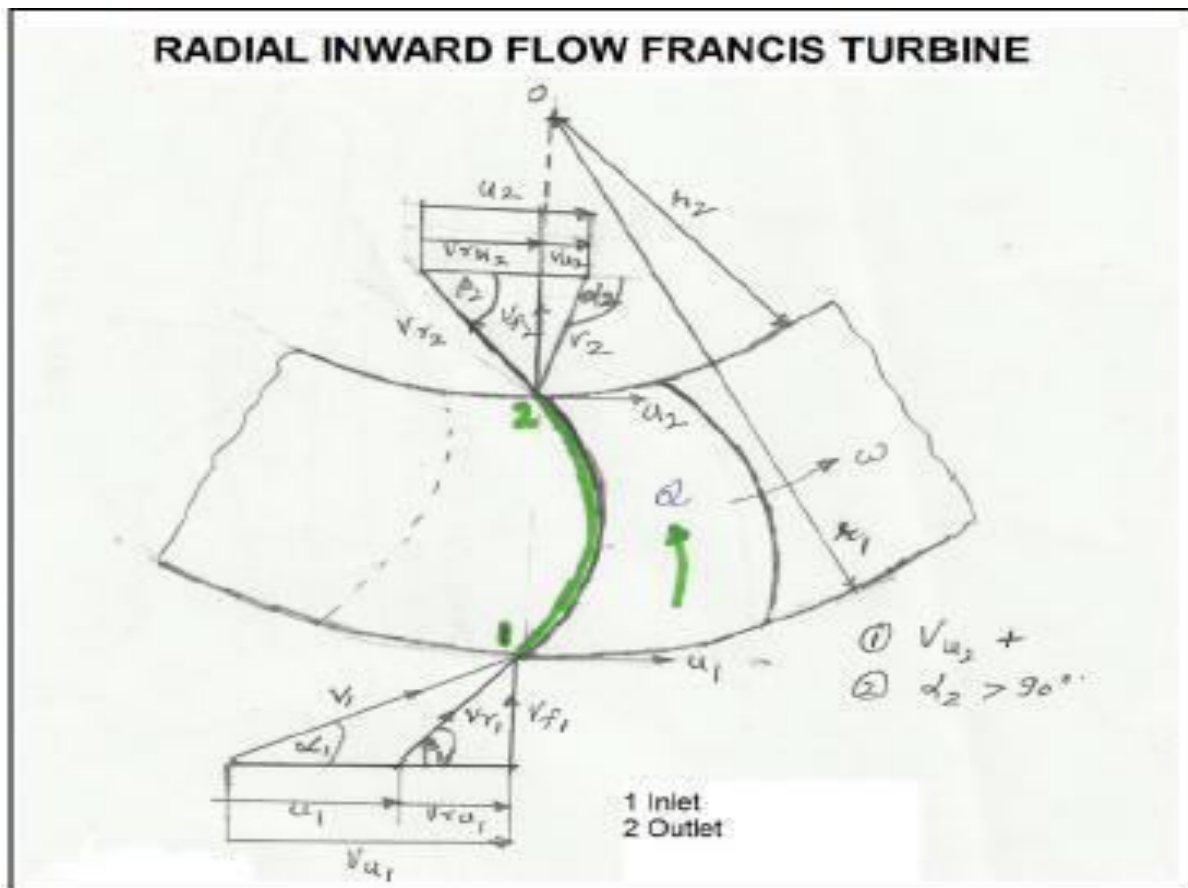
$$= [V_{u1}(\omega R_1) - V_{u2}(\omega R_2)]$$

$$= \left[V_{u1} \left(\frac{2\pi N}{60} R_1 \right) - V_{u2} \left(\frac{2\pi N}{60} R_2 \right) \right]$$



$$W = \left[\frac{V_{u1}\pi D_1 N}{60} - \frac{V_{u2}\pi D_2 N}{60} \right]$$

$$W = [V_{u1}u_1 - V_{u2}u_2] \text{ --- 8}$$





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From inlet velocity triangle

$$V_{f1}^2 = V_1^2 - V_{u1}^2$$

$$V_{r1}^2 = V_{f1}^2 + V_{ru1}^2$$

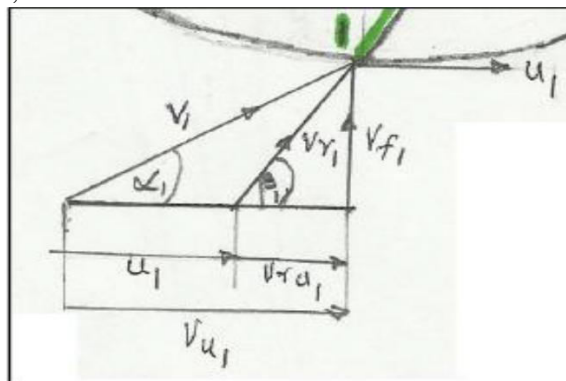
$$V_{r1}^2 = V_1^2 - V_{u1}^2 + (V_{u1} - U_1)^2$$

$$= V_1^2 - \cancel{V_{u1}^2} + \cancel{V_{u1}^2} - 2V_{u1}U_1 + U_1^2$$

Rearranging

$$2V_{u1}U_1 = V_1^2 + U_1^2 - V_{r1}^2$$

$$V_{u1}U_1 = \frac{V_1^2 + U_1^2 - V_{r1}^2}{2} \text{ m}^2/\text{s}^2 \text{ OR Nm/kg... (1)}$$



From outlet velocity triangle

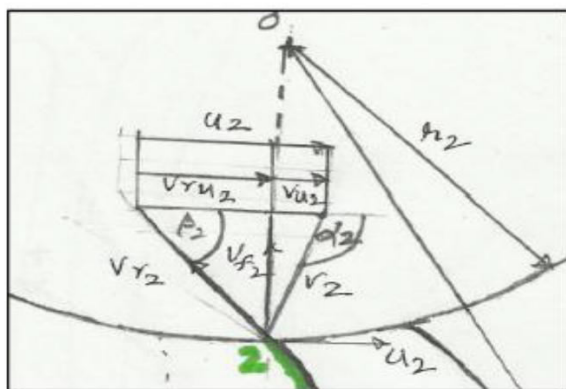
$$V_{r2}^2 = V_{u2}^2 + V_{f2}^2$$

$$= (U_2 - V_{u2})^2 + (V_2^2 - V_{u2}^2)$$

Taking $V_{ru2} = (U_2 - V_{u2})$ in magnitude only and not in directions

$$V_{r2}^2 = U_2^2 - 2V_{u2}U_2 + \cancel{V_{u2}^2} + V_2^2 - \cancel{V_{u2}^2}$$

$$\therefore V_{u2}U_2 = \frac{V_2^2 + U_2^2 - V_{r2}^2}{2} \text{ m}^2/\text{s}^2 \text{ OR Nm/kg... (2)}$$



CASE 1:

Taking direction of rotation as positive

V_{u1} +ve and V_{u2} also +ve.

Work done/kg or Energy transfer in Turbine

$$\text{Work done/kg} = (V_{u1}U_1 - V_{u2}U_2)$$

$$\text{Energy Transfer (E)} = \left[\frac{V_1^2 + U_1^2 - V_{r1}^2}{2} \right] - \left[\frac{V_2^2 + U_2^2 - V_{r2}^2}{2} \right]$$

$$= \left[\frac{V_1^2 - V_2^2}{2} \right] + \left[\frac{U_1^2 - U_2^2}{2} \right] + \left[\frac{V_{r2}^2 - V_{r1}^2}{2} \right]$$

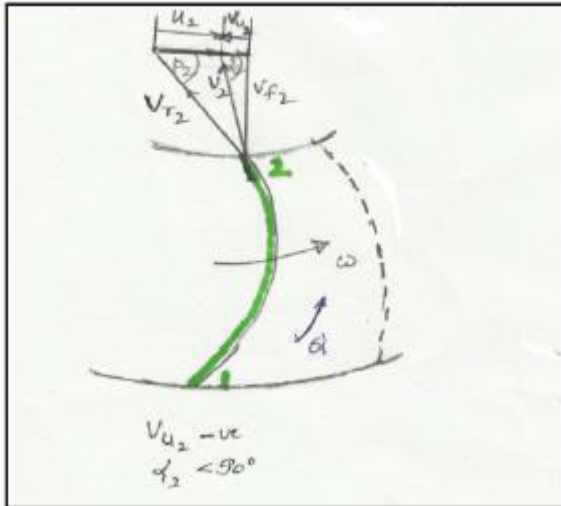


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Components of energy transfer

1. $\frac{V_1^2 - V_2^2}{2}$ is change in absolute kinetic energy in m^2/s^2 or Nm/kg
2. $\frac{U_1^2 - U_2^2}{2}$ is change in centrifugal energy of fluid felt as static pressure change in rotor blades in m^2/s^2 or Nm/kg
3. $\frac{V_{r2}^2 - V_{r1}^2}{2}$ is change in relative velocity energy felt as static pressure change in rotor blades in m^2/s^2 or Nm/kg



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Impulse

1. All potential energies are converted into KE by nozzle before entering to turbine runner
2. Flow regulation is possible without ~~gates~~
3. Flow is regulated by means of a needle valve fitted into the nozzle
4. water may be allowed to enter a part or whole of the wheel circumference
5. wheel does not run full and air has free access to the buckets
6. unit is installed above the tailrace
7. Blades are only in action when they are in front of nozzle

Reaction

- only a portion of the fluid energy is transferred into KE before the fluid enters the turbine.
- Flow regulation is possible with loss
- Flow is regulated by means of a guide-vane assembly
- water is admitted over the circumference of the wheel
- water completely fills the vane passages throughout the operation of turbine.
- unit is kept entirely submerged in water below tailrace.
- blades are in action at all the time.



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Tangential flow turbine:

water flows along the tangent to the path of the runner.

Ex: Pelton wheel.

Radial flow turbine:

In a radial flow turbine, water flows along the radial direction and mainly in the plane normal to the axis of rotation, as it passes through the runner.

It may be either inward radial flow type or outward radial flow type.

Inward radial flow:

water enters at the outer circumference and flows radially inwards to the centre of the runner.

Ex: old Francis turbine, Thomson turbine.

outward radial flow:

water enters at the centre and flows radially outwards towards the outer periphery of the runner.

Ex: Fourneyron turbine.

Mixed Flow turbine

In a mixed flow turbine, the water enters the blades radially and comes out axially and parallel to the turbine shaft.



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Ex: modern Francis turbine

In our subject point of view, the following turbines are important and will be discussed one by one.

1. Pelton wheel
2. Francis turbine
3. Kaplan turbine

Axial Flow Turbine:

Water flows parallel to the axis of the turbine shaft

Eg. 1) Kaplan Turbine, 2) Propeller Turbine.

V - Velocity of Jet (absolute)

v_w - whirl velocity

v_r - Relative velocity (when there is no friction)

u - Vane velocity (Runner)

ϕ - Angle of blade at outlet or Vane angle outlet

β - Angle made by v_r with the direction of motion of the vane at outlet.

v_f - velocity of flow

d - dia of Jet

D - dia of wheel

D^* - dia of penstock



Question and answers: -

Draw velocity triangles at inlet and outlet of typical Francis turbine vane.

Ans. There are three types of velocity triangles for inlet and outlet in Francis turbine. Triangles are made for slow runner, medium runner and fast runner.

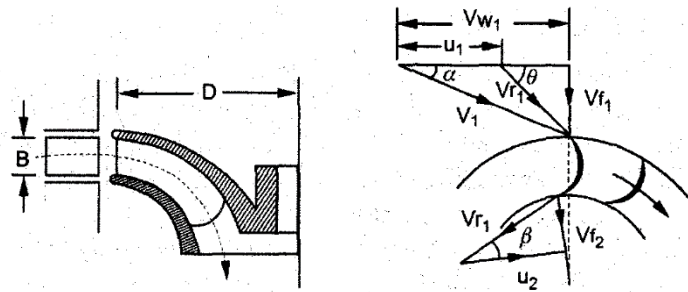


Fig. Slow runner

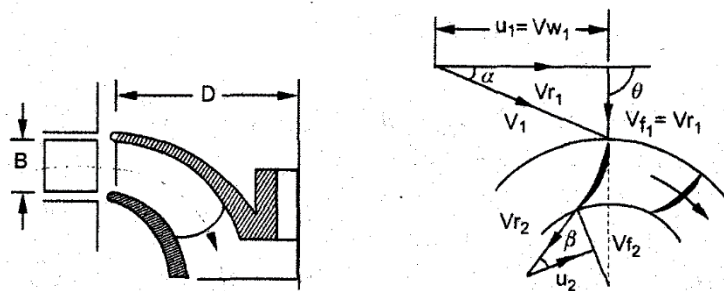


Fig. Medium Runner

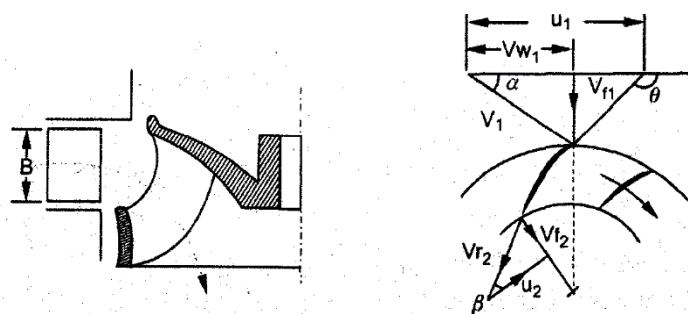


Fig. Fast Runner