## SNS COLLEGE OF TECHNOLOGY

(An autonomous Institution)

Approved by AICTE and Affiliated to Anna University, recognized by UGC
Accredited by NBA-AICTE, NAAC-UGC with 'A++' Grade
Sathy Main Road, SNS Kalvi Nagar, Saravanampatti Post, Coimbatore 641 035, Tamil Nadu, India.

## DEPARTMENT OF MECHANICAL ENGINEERING

# 19MEB 201 - FLUID MECHANICS AND MACHINERY 

Name ..... :
Register Number

:
$\qquad$
Year \& Semester ..... :
Branch / Class :
$\qquad$
Academic Year :

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NAME:.........................................................

REGISTER NO: $\qquad$

Certified that this is the Bonafide record of work done by the above student of the 19MEB201- FLUID MECHANICS AND MACHINERY LABORATORY during the year
$\qquad$ to $\qquad$
$\qquad$

Index

| Ex.No. | Date | Experiment | Page No. | Performance <br> (40) | Viva <br> (10) | Total marks (50) | Staff <br> Initial |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1. |  | Verification of Bernoulli's equation. |  |  |  |  |  |
| 2. |  | Determination of the Coefficient of the Orifice meter. |  |  |  |  |  |
| 3. |  | Determination of the Coefficient of the Venturimeter. |  |  |  |  |  |
| 4. |  | Determination of friction factor for a given set of pipes. |  |  |  |  |  |
| 5. |  | Determination of major and minor losses in pipes |  |  |  |  |  |
| 6. |  | Performance test on centrifugal pump |  |  |  |  |  |
| 7. |  | Performance test on reciprocating pump |  |  |  |  |  |
| 8. |  | Performance studies on Francis turbine |  |  |  |  |  |
| 9. |  | Performance studies on Pelton wheel |  |  |  |  |  |
| 10. |  | Performance studies on Kaplan turbine |  |  |  |  |  |
| Average: |  |  |  |  |  |  |  |


| 19MEB201 | FLUID MECHANICS AND MACHINERY | L | T | P | C |
| :--- | :---: | :---: | :---: | :---: | :---: |
|  |  | 3 | 0 | 2 | 4 |


\section*{| UNIT-I | FLUID PROPERTIES AND FLOW CHARACTERISTICS | $\mathbf{9 + 6}$ |
| :--- | :--- | :--- |}

Units and dimensions - Properties of fluids - mass density, specific weight, specific volume, specific gravity, viscosity, compressibility, vapour pressure, surface tension and capillarity. Flow characteristics -concept of control volume - application of continuity equation, energy equation and momentum equation.

## Lab Experiments:

> Verification of Bernoulli's equation
$>$ Determination of the coefficient of discharge of given Orifice meter/Venturimeter.

```
UNIT-II 
Hydraulic and energy gradient - Laminar flow through circular conduits - Boundary layer concepts -types
of boundary layer thickness -Darcy Weisbach equation -friction factor - Moody diagram - commercial
pipes - minor losses -Flow through pipes in series and parallel.
Lab Experiments:
    Determination of friction factor for a given set of pipes
    Determination of major and minor losses in pipes
```

| UNIT-III | DIMENSIONAL ANALYSIS AND SIMILITUDE | $\mathbf{9}$ |
| :--- | :--- | :--- |

Need for dimensional analysis -dimensional analysis by using Buckingham's $\pi$ theorem method Similitude -types of similitude - Dimensionless parameters - application of dimensionless Parameters-Model analysis.

| UNIT-IV | PUMPS | $\mathbf{9 + 6}$ |
| :--- | :--- | :--- |
| Impact of jets -Euler's equation - Theory of roto - dynamic machines- various efficiencies-velocity |  |  |
| components at entry and exit of the rotor - velocity triangles -Centrifugal pumps-working principle |  |  |
| - work done by the impeller - Reciprocating pump - working principle. |  |  |
| Lab Experiments: |  |  |
| $>$ Performance studies on centrifugal pump |  |  |
| $>$ Performance studies on reciprocating pump |  |  |


| UNIT-V | TURBINES | $\mathbf{9 + 1 2}$ |
| :--- | :--- | :--- |

Classification of turbines -heads and efficiencies -velocity triangles. Axial, radial and mixed flow turbines. Pelton wheel, Francis turbine and Kaplan turbines - working principles - work done by water on the runner -draft tube. Specific speed - unit quantities.
Lab Experiments:
> Performance studies on Pelton wheel
$>$ Performance studies on Francis turbine
> Performance studies on of Kaplan turbine

| L:45 | P:30 | T: 75 PERIODS |
| :--- | :--- | :--- |

## TEXT BOOKS

1. Yunus A. C̦engel, John M. Cimbala., Fluid Mechanics: Fundamentals and Applications, McGraw - Hill Higher Education, 2010, $3^{\text {rd }}$ edition (Unit I, II, III, IV, V)
2. Bansal, R.K., Fluid Mechanics and Hydraulics Machines, Laxmi Publications (P) Ltd., New Delhi. 2011. 10 ${ }^{\text {th }}$ Edition (Unit I, II, III, IV, V)

## REFERENCES

1. Modi P.N. and Seth, S.M. "Hydraulics and Fluid Mechanics including Hydraulic Machines", Standard Book House, New Delhi 2013. 19thEditon (Unit I, II, III, IV, V)
2. Robert W. Fox, Alan T. McDonald, Philip J. Pritchard, "Fluid Mechanics and Machinery", 2011. (UNIT - I, II, III, IV, V)
3. Kumar. K.L., Engineering Fluid Mechanics, Eurasia Publishing House (P) Ltd., New Delhi, 2010. 8th Edison (Unit I, II, III)
4. Streeter. V. L., and Wylie, E.B., Fluid Mechanics, McGraw Hill, 2010. 9th Edition (Unit I, II, III)
5. Rajput. R. K, "A text book of Fluid Mechanics and Hydraulic Machines", S. Chand \& Company Ltd., New Delhi, sixth edition, 2010 (Unit I, II, III, IV, V).

## WEB RESOURCES <br> https://nptel.ac.in/courses/105101082/ <br> https://www.youtube.com/watch?v=fa0zHI6nLUo\&list=PLbMVogVi5nJTZJHsH6uLCOOOI-ffGyBEm https://nptel.ac.in/courses/112105183/

## COURSE OUTCOMES

At the end of the course students should be able to

- Explain the fundamental concepts of fluid mechanics with different properties of fluids.
- Analyse and calculate major and minor losses associated with pipe flow in piping networks.
- Predict the nature of physical quantities and to predict the behavior of the prototype/model by applying model laws.
- Analyse the performance of pumps.
- Analyse the performance of hydraulic turbines.

Mapping Course outcomes with Program Outcomes (POs) and Program Specific Outcomes(PSOs)

| COURSE <br> OUTCOMES | PROGRAMME OUTCOMES |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  | PSO1 | PSO2 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | a | b | c | d | e | f | g | h | i | j | k | l |  |  |  |  |  |  |  |  |
| I | S | S |  |  |  |  |  |  |  |  |  |  | S |  |  |  |  |  |  |  |
| II | S | S |  |  |  | M |  |  |  |  |  |  | S |  |  |  |  |  |  |  |
| III | S | S | S |  |  | M |  |  |  |  |  |  | S | M |  |  |  |  |  |  |
| IV | S | S |  |  | S |  |  |  |  |  |  |  | S | S |  |  |  |  |  |  |
| V | S | S |  |  | S |  |  |  |  |  |  |  | M | S |  |  |  |  |  |  |

## SNS COLLEGE OF TECHNOLOGY- COIMBATORE 35

(An Autonomous Institution)
COIMBATORE -35

## DEPARTMENT OF MECHANICAL ENGINEERING

## Vision

The Department of Mechanical Engineering Endeavors to be a recognized Knowledge Centre for Quality Education and Research by Producing Competent and Creative Engineers to meet the Global Challenges.

## Mission

$>$ To Impart Knowledge by Providing Effective Means of Learning and Facilitate Personal Commitment for Productive Careers Globally.
$>$ To Provide Students and Faculty Members with Opportunitiesfor Creative Thinking, Interpretation, Application andKnowledge Dissemination.
$>$ To Inculcate Life-Long Learning, Human Values andEntrepreneurship Culture in the fields of Engineering Design,Thermal, Manufacturing, Automation etc.

To Link with Industries and Institutions for Mutual Benefit.
Program Educational Objectives (PEOs) : B.E-Mechanical Engineering
PEO 1:Graduate will be Capable of Applying the Skills of Basic Science, Mathematics, and Engineering for their Successful Professional Career.

PEO 2:Graduate will Design and Conduct Experiments, Interpret, Analyze and Communicate Effectively for the Real Life Problems.

PEO 3:Graduate will apply the State of the Art Tools, Technologies and Interdisciplinary Knowledge to Bring Enhancements in the Society.

PEO 4:Graduate will possess the Qualities of Self-Learning, Team Work, Ethical Conduct and Entrepreneurial Spirit.

COIMBATORE -35
DEPARTMENT OF MECHANICAL ENGINEERING
PROGRAMME OUTCOMES (PO)
$\left.\begin{array}{|l|l|l|}\hline \text { PO-1 } & \begin{array}{l}\text { Engineering } \\ \text { Knowledge }\end{array} & \text { Problem Analysis } \\ \text { PO-2 } & \begin{array}{l}\text { Apply the knowledge of mathematics, science, engineering fundamentals, and an engineering } \\ \text { specialization to the solution of complex engineering problems. }\end{array} \\ \hline \text { PO-3 } & \begin{array}{l}\text { Development of } \\ \text { solutions }\end{array} & \begin{array}{l}\text { Identify, formulate, review research literature, and analyze complex engineering problems reaching } \\ \text { substantiated conclusions using first principles of mathematics, natural sciences, and engineering } \\ \text { sciences. }\end{array} \\ \hline \text { PO-4 } & \begin{array}{l}\text { Conduct Investigations } \\ \text { of complex problems }\end{array} & \begin{array}{l}\text { Design solutions for complex engineering problems and design system components or processes that } \\ \text { meet the specified needs with appropriate consideration for the public health and safety, and the } \\ \text { cultural, societal, and environmental considerations. } \\ \text { interpretation of data, and synthesis of the information to provide valid conclusions. }\end{array} \\ \hline \text { PO-5 } & \text { Modern Tool usage } & \begin{array}{l}\text { Create, select, and apply appropriate techniques, resources, and modern engineering and IT tools } \\ \text { including prediction and modeling to complex engineering activities with an understanding of the } \\ \text { limitations }\end{array} \\ \hline \text { PO-6 } & \begin{array}{l}\text { The Engineer and } \\ \text { Society }\end{array} & \begin{array}{l}\text { Apply reasoning informed by the contextual knowledge to assess societal, health, safety, legal and } \\ \text { cultural issues and the consequent responsibilities relevant to the professional engineering practice }\end{array} \\ \hline \text { PO-7 } & \begin{array}{l}\text { Environment and } \\ \text { Sustainability }\end{array} & \begin{array}{l}\text { Understand the impact of the professional engineering solutions in societal and environmental } \\ \text { contexts, and demonstrate the knowledge of, and need for sustainable development }\end{array} \\ \hline \text { PO-8 } & \text { Ethics } & \begin{array}{l}\text { Apply ethical principles and commit to professional ethics and responsibilities and norms of the } \\ \text { engineering practice }\end{array} \\ \hline \text { PO-9 } & \begin{array}{l}\text { Individual and Team } \\ \text { Work }\end{array} & \begin{array}{l}\text { Function effectively as an individual, and as a member or leader in diverse teams, and in } \\ \text { multidisciplinary settings }\end{array} \\ \hline \text { PO-10 } & \text { Communication } & \begin{array}{l}\text { Communicate effectively on complex engineering activities with the engineering community and with } \\ \text { society at large, such as, being able to comprehend and write effective reports and design } \\ \text { documentation, make effective presentations, and give and receive clear instructions. } \\ \text { and finance }\end{array} \\ \hline \text { Life-long learning } & \begin{array}{l}\text { Recognize the need for, and have the preparation and ability to engage in } \\ \text { independent and life-long learning in the broadest context of technological change. }\end{array} \\ \hline \text { maltidisciplinary environments. a member and leader in a team, to manage projects and in } \\ \text { these to one' owdge and understanding of the engineering and management principles and appy }\end{array}\right\}$

At the end of the program, graduate will be able to

## PROGRAM SPECIFIC OUTCOMES (PSOS)

PSO-1 Apply engineering knowledge and design \& analysis tools to solve problems in the domains of structural, thermal and fluid mechanics.

PSO-2 Engage professionally in industries or as an entrepreneur by applying manufacturing and management practices

## INTRODUCTION

## Fluid mechanics and Machinery:

Fluid mechanics is a fascinating study of fluids at its rest position or in its motion. A fluid is defined as a substance that continually deforms (flows) under an applied shear stress regardless of the magnitude of the applied stress. Fluid mechanics have played an important role in human life. Therefore, it also attracted many curious people. Imagine a substance that can change its form according to the container that holds it and has unharnessed potential to cut through metals, lift heavy loads and yet is feeble. On the other hand, fluid machines are devices that work with fluids or work on fluids to produce desired effect or work. These machines are used to transfer fluids from one level to the other, extract energy from the fluid and even to measure the properties of fluid.

## Fluid mechanics and Machinery Lab:

Fluid mechanics lab is a place of experimentation and excitement where the fluid, mainly water is used for study. The entire lab is split into three major sections based on the property of the fluid to store energy, release energy and the hydraulic devices used for it. They are:

- Energy consuming devices - pumps
- Energy producing devices - Turbines
- Energy measuring devices - Meters

The first section deals with the energy consuming devices such aspumps which converts electrical energy into flow energy to energize the fluid to lift it from a lower datum to a higher datum. The second section consist of devices that extracts the flow energy [resent in the fluid and converts into electrical energy. The third section deals with the devices which are basically used to measure the amount of fluid flow rate and different meters are used for different working condition of water based on topography, volume and the energy level of water.

## COURSE OBJECTIVE:

- To demonstrate the Bernoulli's equation in Bernoulli's apparatus.
- To measure the fluid flow through Orifice meter, venturimeter.
- To evaluate the friction factor for a given set of pipes.
- To predict the performance parameter of various rotary and reciprocating pumps.
- To estimate the performance parameter of various rotary turbines.


## COURSE CONTENT:

| PART A - ENERGY MEASURING DEVICES |  |  |  |
| :---: | :--- | :---: | :---: |
| S.NO. | EXPERIMENTS |  |  |
| 1 | Verification of Bernoulli's equation. |  |  |
| 2 | Determination of the Coefficient of the Orifice meter |  |  |
| 3 | Determination of the Coefficient of the Venturimeter meter |  |  |
| 4 | Determination of friction factor for a given set of pipes. |  |  |
| 5 | Determination major and minor loss in pipes. |  |  |
| 6 | Performance test on centrifugal pump |  |  |
| 7 | Performance test on reciprocating pump |  |  |
| PART C - ENERGY PRODUCING DEVICES |  |  |  |
| 8 | Performance test on Francis turbine |  |  |
| 9 | Performance test on Pelton wheel turbine |  |  |
| 10 | Performance test on Kaplan turbine |  |  |

## SAFETY INSTRUCTIONS:

- Read and fully comprehend the lab procedure as set forth in the lab manual before you begin any experiment. If you do not understand the procedure, contact the staff incharge.
- All lab experiments require a closed toe shows. No open toed shoes; no loose fitting clothing; Jewelry should be removed; long hair should be tied back
- Always maintain awareness of the surrounding activities and walk in aisles to the extent possible.
- Maintain clean and orderly laboratories and work area. Discard immediately unwanted items. Leave coats, books and note books in the lecture room. Make sure all spilled liquids are wiped up immediately.
- Students are responsible for maintaining work area in a safe and reasonable condition.
- Be aware of the various experiment controls (start button, stop button, speed control) for each lab.
- Be aware of the experiment harness when conducting experiments.
- Wear safety eyewear when needed. Additional safety equipment must be utilized based on specific experiment requirements.
- Do not leave experiments running unattended.
- Immediately report any spills, equipment malfunctions, injuries or other perceived safety hazards to your staff member.
- Work deliberately and carefully.
- Make note of fire escape routes and emergency locations.


## EQUIPMENTS:

- Bernoulli's apparatus
- Orifice meter setup
- Venturi meter setup
- Major and minor losses measuring setup
- Friction factor apparatus setup pipes
- Centrifugal pump
- Reciprocating pump setup
- Pelton wheel setup
- Francis turbine setup
- Kaplan turbine setup

EX.NO:1
VERIFICATION OF BERNOULIS EQUATION
DATE :

## PRIOR LEARNING:

1. Continuity equation
2. Euler's equation
3. Bernoulli's equation

## OBJECTIVE:

To verify the Bernoulli's theorem.

## THEORY:

Bernoulli's theorem states that when there is a continues connection between the particle of flowing mass liquid, the total energy of any sector of flow will remain same provided there is no reduction or addition at any point. i.e. sum of pressure head and velocity head is constant.


Bernoulli's apparatus

## Write the physical description of the following:

1. Hydraulic bench
2. Piezometer
3. Venturimeter
4. Series of tubes in the se3t up

## APPARATUS REQUIRED:

1. Bernoulli's apparatus with equipment
2. Hydraulic bench
3. Piezometer
4. Venturimeter

## FORMULAE:

## 1. Actual Discharge $\left(Q_{a}\right)$ :

$=[($ Cross sectional area of the tank) X (Rise of water level in the tank )] / (Time taken to rise the water)
$\mathrm{Q}_{\mathrm{a}}=\mathrm{AH} / \mathrm{t}$ in $\mathrm{m}^{3} / \mathrm{s}$
Where,
A = Area of the tank in $\mathrm{m}^{2}$
$\mathrm{H}=$ Height of water rise for the given time
' $t$ second' in $m$, in piezometer
$\mathrm{t}=$ Time taken for ' H ' m rise of water in s
2. Suction Velocity $\left(\mathbf{V}_{\mathrm{x}}\right)$ :
$\underline{\mathbf{V}}_{\underline{x}}=\mathbf{V} \mathrm{XQ} / \mathrm{a}_{\mathrm{x}}$
$\mathrm{Q}=$ Actual discharge in $\mathrm{m}^{3} / \mathrm{s}$
$\mathrm{a}_{\mathrm{x}}=$ Cross section area of pipe in $\mathrm{m}^{2}$

## 3. Velocity head $\left(V_{h}\right)$ :

$$
\mathrm{V}_{\mathrm{h}}=100 \mathrm{X} \mathrm{~V}_{\mathbf{x}}{ }^{2} / \mathbf{2 g}
$$

## WORK SPACE:

## WORKING PROCEDURE

| Sl.no. | Cross Section area | Piezometer reading $\mathbf{p} \times \mathbf{c m}$ of water |  |  | Suction velocity (V) | $\begin{aligned} & \text { Velocity } \\ & \text { head } \\ & 100 \times \\ & \left(\mathrm{v}^{2} \times 2 \mathrm{~g}\right) \end{aligned}$ | Pressure <br> head + <br> Velocity <br> head |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Unit / Run values | $\mathrm{m}^{2}$ | T1 50 | T1 40 | T3 30 | m/sec. | cm of water | cm of water |
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CALCULATION

| S. No. | Expression | Formula/value | Unit |
| :---: | :--- | :--- | :---: |
| 1 | Piezometer Reading |  |  |
| 2 | Time for 5 cm rise |  |  |
| 3 | Discharge Q in |  |  |
| 4 | Pressure Head |  |  |
| 5 | Velocity Head |  |  |
| 6 | Datum head |  |  |
| 7 | Total Head |  |  |
| NOTE: |  |  |  |
| 1 mm of Hg = 13.6/1000 m of $\mathrm{H}_{2} \mathrm{O}$ |  |  |  |
| $1 \mathrm{~kg} / \mathrm{cm}^{2}=10.33 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$ |  |  |  |

## WORKING PROCEDURE:

## VIVA:

1. Write Bernoulli's equation.
2. What are assumptions of Bernoulli's equation?
3. Write Euler's equation.
4. Explain about a coefficient of discharge of nozzle.
5. What is pitot static tube, and piezometer?

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at te beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## RESULT:

Verification of Bernoulli's theorem was made.

EX.NO:2
DATE :

## DETERMINATION OF THE COEFFICIENT OF THE ORIFICEMETER

## PRIOR LEARNING:

1. Bernoulli's principle
2. Continuity equation
3. Vena-contracta

## OBJECTIVE:

To determine the co-efficient of discharge through the orifice meter.

## THEORY:

An orifice meter is a conduit and a restriction to create a pressure drop. An hour glass is a form of orifice. A nozzle, venturi or thin sharp edged orifice can be used as the flow restriction. The principle of the orifice meter is identical with that of the venturi meter. The reduction of the cross section of the flowing stream in passing through the orifice increases the velocity head at the expense of the pressure head, and the reduction in pressure between the taps is measured by a manometer. Bernoulli's equation provides a basis for correlating the increase in velocity head with the decrease in pressure head.


Orifice meter Setup

## Write the physical description of the following:

1. Flange:
2. Vena-contracta:
3. Pipe:

## APPARATUS REQUIRED:

1. Orifice meter set-up.
2. Stop watch.
3. Meter scale.
4. Collecting tank fitted with piezometer.
5. Differential U-tube mercury manometer.

## FORMULAE:

$\mathrm{X}_{1}=$ Level of Mercury in left limb
$\mathrm{X}_{2}=$ Level of Mercury in right limb
$X=X_{1}-X_{2} m$ of Mercury $(H g)$
$\mathrm{h}=\mathrm{X}\left(\mathrm{S}_{\mathrm{Hg}} / \mathrm{S}_{\text {water }}-1\right) \mathrm{m}$ of water
$=\mathrm{X}(13.6 / 1-1)$
$=\mathrm{X}(12.6-1)$
$\mathrm{S}_{\mathrm{Hg}} \mathrm{Sp}$. gravity of Hg
$S_{\text {water }} S p$. gravity of water
$\mathbf{Q}_{\mathrm{T}}=\mathbf{a}_{1} \mathbf{a}_{\mathbf{2}} \sqrt{ }(\mathbf{2 g h}) / \sqrt{ }\left(\mathbf{a}_{1}{ }^{\mathbf{2}} \cdot \mathbf{a}_{2}{ }^{\mathbf{2}}\right) \mathrm{in} \mathrm{m}^{3} / \mathrm{s}$ where,
$\mathrm{Q}_{\mathrm{t}}=$ Theoretical discharge, $\mathrm{m}^{3} / \mathrm{s}$
$\mathrm{a}_{1}=$ Area of the main pipe in $\mathrm{m}^{2}$
$\mathrm{a}_{2}=$ Area of the orifice meter in $\mathrm{m}^{2}$
$\mathrm{h}=$ Pressure head difference in m of wate

## Actual discharge ( $\mathbf{Q}_{\mathrm{a}}$ ),

$=[($ Cross sectional area of the tank ) $\mathbf{X}$ (Rise of water level in the tank )] / (Time taken to rise the water)
$\mathrm{Q}_{\mathrm{a}}=\mathrm{AH} / \mathrm{t} \mathrm{m}^{3} / \mathrm{s}$
$\mathrm{Q}_{\text {act }}=$ $\qquad$

Coefficient of Discharge, $C_{d}=Q_{a} / Q_{t}$

| TABULATION |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Diameter of the inlet ( $\mathrm{d}_{1}$ ) |  |  |  |  |  |  |  |
| Diameter of the orifice ( $\mathrm{d}_{2}$ ) |  |  |  |  |  |  |  |
| Dimension of the Collecting tank |  |  |  |  |  |  |  |
| Length |  |  |  |  |  |  |  |
| Breadth = |  |  |  |  |  |  |  |
| S.No | Manometer reading |  | $\begin{aligned} & \text { Manometer } \\ & \text { head } h=\left(x_{1} \sim x_{2}\right) \\ & \times 12.6 \times 10^{-2} \\ & \mathrm{~m} \end{aligned}$ | Time taken for $\mathrm{H}=10$ cm rise of water t Sec | Actual discharge$\begin{aligned} & \text { Qact }=\mathbf{A H} / \\ & \mathbf{t} \quad\left(\mathrm{m}^{3} / \mathrm{s}\right) \end{aligned}$ | Theoretical discharge Qth $\mathrm{m}^{3} / \mathrm{s}$ | Co-efficient of discharge $\mathbf{C}_{\mathrm{d}}$ (no unit) |
|  | $\mathrm{x}_{1} \mathrm{~cm} \text { of }$ Hg | $\begin{aligned} & \mathrm{x}_{2} \mathrm{~cm} \text { of } \\ & \mathbf{H g} \end{aligned}$ |  |  |  |  |  |
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|  |  |  |  |  |  | $\mathrm{Cd}=$ |  |

CALCULATION

| S. No. | Expression | Formula/value | Unit |
| :---: | :--- | :--- | :--- |
| 1 | Diameter of inlet, $\mathrm{d}_{1}$ |  |  |
| 2 | Diameter of orifice meter, $\mathrm{d}_{2}$ |  |  |
| 3 | Specific gravity of the mercury, $\mathrm{S}_{\mathrm{m}}$ |  |  |

## EXPERIMENTAL PROCEDURE:

MODEL GRAPH:
Actual Discharge $\left(\mathrm{Q}_{\mathrm{a}}\right)$ VS Theoretical Discharge $\left(\mathrm{Q}_{\mathrm{t}}\right)$

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at te beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## RESULT:

The mean value of Co-efficient of discharge of orifice meter, $\mathrm{C}_{\mathrm{d}}=$

EX.NO:3
DATE :

## DETERMINATION OF THE COEFFICIENT OF THE VENTURIMETER

## PRIOR LEARNING:

1. Bernoulli's principle
2. Veturi effect
3. Continuity equation

## OBJECTIVE:

To determine the co-efficient of discharge of the given venturimeter.

## THEORY:

Venturimeter is a device used to measure the flow rate or discharge of fluid through a pipe. Venturimeter is an application of Bernoulli's equation. Total energy of the flowing fluid at any point in its flow is same. The principle of venture meter is firstly developed by G.B. Venturi in 1797. The principle is that when cross sectional area of the flow is reduced then a pressure difference is created between the different areas of flow which helps in measuring the difference in pressure. With the help of this pressure difference we can easily measure the discharge in flow by using Bernoulli's equation.


Venturimeter Setup
Write the physical description of the following:

1. Short converging part:
2. Throat:

## 3. Diverging part:

## APPARATUS REQUIRED:

1. Venturimeter set-up.
2. Stop watch.
3. Meter scale.
4. Collecting tank fitted with piezometer.
5. Differential U-tube mercury manometer.

## WORK SPACE:

$\mathbf{Q}_{\mathrm{T}}=\mathbf{a}_{1} \mathbf{a}_{\mathbf{2}} \sqrt{ }(\mathbf{2 g h}) / \sqrt{ }\left(\mathbf{a}_{1}{ }^{2}-\mathbf{a}_{2}{ }^{\mathbf{2}}\right) \mathrm{in} \mathrm{m}^{3} / \mathrm{s}$ where,
$\mathbf{Q}_{\mathbf{t}}=$ Theoretical discharge, $\mathbf{m}^{3} / \mathbf{s}$
$\mathrm{a}_{1}=$ Area of the venturimeter inlet in $\mathrm{m}^{2}$
$\mathrm{a}_{2}=$ Area of the venturimeter throat in $\mathrm{m}^{2}$
$h=$ Pressure head difference in $m$ of water

## Actual discharge ( $\mathrm{Q}_{\mathrm{a}}$ ),

$=[($ Cross sectional area of the tank ) X (Rise of water level in the tank )] / (Time taken to rise the water)
$Q_{a}=A H / t \mathrm{~m}^{3} / \mathrm{s}$
$Q_{a c t}=$ $\qquad$
Coefficient of Discharge, $C_{d}=Q_{a} / Q_{t}$

WORKING PROCEDURE:

CALCULATION:


| S. No. | Expression | Formula/value | Unit |
| :---: | :---: | :---: | :---: |
| 1 | Diameter of inlet, $\mathrm{d}_{1}$ |  |  |
| 2 | Diameter of throat, $\mathrm{d}_{2}$ |  |  |
| 3 | Specific gravity of the mercury, $\mathrm{S}_{\mathrm{Hg}}$ |  |  |
| 4 | Specific gravity of the liquid, $S_{\text {water }}$ |  |  |
| 5 | Difference in manometer level, X |  |  |
| 6 | Differential head, h |  |  |
| 7 | Area of inlet, $a_{1}$ |  |  |
| 8 | Area of throat, $\mathrm{a}_{2}$ |  |  |
| 9 | Theoretical discharge, $\mathrm{Q}_{\mathrm{t}}$ |  |  |
| 10 | Actual discharge, Qact |  |  |
| 11 | Coefficient of discharge |  |  |

## GRAPH:

Actual Discharge ( $\mathrm{Q}_{\mathrm{a}}$ ) Vs Theoretical Discharge $\left(\mathrm{Q}_{\mathrm{t}}\right)$

## Qt <br>  <br> $\mathrm{Cd}=\mathrm{dQa} / \mathrm{dOt}$ <br> Qa

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at the beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## RESULT:

The mean value of Co-efficient of discharge of the given venturimeter, $\mathrm{Cd}=$

EX.NO :4
DATE :

DETERMINATION OF THE FRICTION FACTOR FOR A GIVEN
SET OF PIPES

## PRIOR LEARNING:

Various losses in fluid flow
Darcy-Weisbach equation

## OBJECTIVE:

To determine the friction factor ' f ' for the given pipe.

## THEORY:

Energy input to the gas or liquid is needed to make it flow through the pipe or conduit. This energy input is needed because there is frictional energy loss (also called frictional head loss or frictional pressure drop) due to the friction between the fluid and the pipe wall and internal friction within the fluid.


Pipe setup for Friction Factor

## Write the physical description of the following:

1. What do you meant by friction and list out its effects?
2. What do you meant by major loss in pipe?
3. Write down the Darcy-Weisbach equation?
4. What are the types of losses in pipe flow?

## APPARATUS REQUIRED:

1. A pipe provided with inlet, outlet and pressure tapping.
2. Differential U-tube manometer.
3. Collecting tank with piezoelectric and control valves.
4. Stop Watch.
5. Meter Scale

FORMULAE:

## 1. Difference in manometer level ( x ):

$\mathrm{x}=\mathrm{h}_{1} \sim \mathrm{~h}_{2}$ in m.
Where,
$\mathrm{h}_{1}=$ Manometer head in one limb of manometer in $m$
$\mathrm{h}_{2}=$ Manometer head in another limb of manometer in $m$

## 2. Differential Head (h):

$h=x\left[\left(S_{m} / S_{1}\right)-1\right]$ in $m$ of water
Where,
$\mathrm{S}_{\mathrm{m}}=$ Specific gravity of the mercury $=13.6$
$S_{1}=$ Specific gravity of the liquid $($ water $)=1$

## 3. Actual Discharge $\left(\mathbf{Q}_{\mathrm{a}}\right)$ :

$\mathrm{Q}_{\mathrm{a}}=\mathrm{AH} / \mathrm{t} \mathrm{in} \mathrm{m}^{3} / \mathrm{s}$
Where,
$\mathrm{A}=$ Area of the collecting tank in $\mathrm{m}^{2}$
$\mathrm{H}=$ Rise of the water in the collecting tank in m $\mathrm{t}=$ Time taken for Hcm rise of water in the tank in s

## 4. Cross sectional area of the pipe (a)

$\mathrm{a}=(\pi / 4) \mathrm{xd}^{2}$ in $\mathrm{m}^{2}$
Where,
$\mathrm{d}=$ Diameter of the pipe in m

## 5. Velocity ( v :

$\mathrm{v}=\mathrm{Q}_{\mathrm{a}} / \mathrm{a}$ in $\mathrm{m} / \mathrm{s}$
Where,
$a=$ Area of the pipe in $\mathrm{m}^{2}$
6. Friction factor (f) :
$\mathrm{f}=\left(2 \mathrm{gdh}_{\mathrm{f}}\right) / \mathrm{Lv}^{2}$
Where,
$\mathrm{h}_{\mathrm{f}}=$ Head loss in m
$L=$ Length of pipe in $m$

## WORK SPACE:

## Calculations

| S. No. | Expression | Formula/value | Unit |
| :---: | :---: | :---: | :---: |
| 1 | $\mathrm{h}_{1}=$ Manometer head in one limb of manometer |  |  |
| 2 | $\mathrm{h}_{2}=$ Manometer head in another limb of manometer |  |  |
| 3 | $\mathrm{Sm}=$ Specific gravity of the mercury $=$ |  |  |
| 4 | $S_{1}=$ Specific gravity of the liquid $($ water $)=$ |  |  |
| 5 | Actual Discharge (Qa ) : |  |  |
| 6 | Velocity (V) : |  |  |
| 7 | Friction factor (f) : |  |  |

## NOTE:

1 mm of $\mathrm{Hg}=13.6 / 1000 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$


## WORKING PROCEDURE

## GRAPH:

Flow rate Vs Head loss


Flow Rate, Q

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at the beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## VIVA:

1. What is the relationship between friction head loss and pipe diameter?
2. What is the relationship between friction head loss and flow velocity?
3. What is the relationship between friction head loss and pipe length?
4. How is the flow rate and head loss related?
5. If flow velocity doubles, what happen to the frictional head loss?

## RESULT:

The mean value of friction factor of given set of pipes is obtained as, $\mathrm{f}=$

EX.NO :5
DATE :

## DETERMINATION OF MINOR LOSSES IN PIPES

## PRIOR LEARNING:

Various losses in fluid flow
Bernoulli's Equation

## OBJECTIVE:

To determine the minor losses for the given pipe.

## THEORY:

Minor losses in pipe flow are a major part in calculating the flow, pressure, or energy reduction in piping systems. Liquid moving through pipes carries momentum and energy due to the forces acting upon it such as pressure and gravity. Just as certain aspects of the system can increase the fluids energy, there are components of the system that act against the fluid and reduce its energy, velocity, or momentum. Friction and minor losses in pipes are major contributing factors.


Write the physical description of the following:
5. Sudden enlargement
6. 'L' Bend
7. Sudden contraction

## APPARATUS REQUIRED:

6. A pipe provided with inlet, outlet and pressure tapping.
7. Differential U-tube manometer.
8. Collecting tank with piezoelectric and control valves.
9. Stop Watch.
10. Meter Scale

## FORMULAE:

7. Difference in manometer level ( x ):
$\mathrm{x}=\mathrm{h}_{1} \sim \mathrm{~h}_{2}$ in m.
Where,
$\mathrm{h}_{1}=$ Manometer head in one limb of manometer in m
$\mathrm{h}_{2}=$ Manometer head in another limb of manometer in $m$

## 8. Differential Head (h):

$h=x\left[\left(S_{m} / S_{l}\right)-1\right]$ in $m$ of water
Where,
$\mathrm{S}_{\mathrm{m}}=$ Specific gravity of the mercury $=13.6$
$\mathrm{S}_{\mathrm{l}}=$ Specific gravity of the liquid $($ water $)=1$

## 9. Actual Discharge ( $\mathrm{Q}_{\mathrm{a}}$ ):

$\mathrm{Q}_{\mathrm{a}}=\mathrm{AH} / \mathrm{t}$ in $\mathrm{m}^{3} / \mathrm{s}$
Where,
$\mathrm{A}=$ Area of the collecting tank in $\mathrm{m}^{2}$
$\mathrm{H}=$ Rise of the water in the collecting tank in m
$\mathrm{t}=$ Time taken for Hcm rise of water in the tank in s

## 10. Cross sectional area of the pipe (a)

$\mathrm{a}=(\pi / 4) \mathrm{xd}^{2}$ in $\mathrm{m}^{2}$
Where,
$\mathrm{d}=$ Diameter of the pipe in m

## 11. Velocity (V) :

$$
\begin{aligned}
& \mathrm{V}_{1}=\mathrm{Q}_{\mathrm{a}} / \mathrm{a}_{1} \text { in } \mathrm{m} / \mathrm{s} \\
& \mathrm{~V}_{2}=\mathrm{Q}_{\mathrm{a}} / \mathrm{a}_{2} \text { in } \mathrm{m} / \mathrm{s}
\end{aligned}
$$

## WORK SPACE:

## For Bend and Elbow

$$
\mathrm{h}=\mathrm{V}^{2} / 2 \mathrm{~g}
$$

12. Sudden Contraction
$\mathrm{h}=\mathrm{V}_{2}{ }^{2} / 2 \mathrm{~g}$
13. Sudden Enlargement:
$\mathrm{h}=\left(\mathrm{V}_{1}-\mathrm{V}_{2}\right)^{2} / 2 \mathrm{~g}$
$V_{1}$ and $V_{2}$ are velocity water at before and after the sudden changes in pipe cross section

## Calculations

| S. No. | Expression | Formula/value | Unit |
| :---: | :---: | :---: | :---: |
| 1 | $\mathrm{h}_{1}=$ Manometer head in one limb of manometer |  |  |
| 2 | $\mathrm{h}_{2}=$ Manometer head in another limb of manometer |  |  |
| 3 | $\mathrm{Sm}=$ Specific gravity of the mercury = |  |  |
| 4 | $\mathrm{S}_{\mathrm{I}}=$ Specific gravity of the liquid $($ water $)=$ |  |  |
| 5 | Actual Discharge (Qa ) |  |  |
| 6 | Velocity (V) : |  |  |
| 7 | Minor losses(f) : |  |  |
| NOTE: |  |  |  |
| 1 mm of $\mathrm{Hg}=13.6 / 1000 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$ |  |  |  |



## WORKING PROCEDURE

GRAPH:
Flow rate Vs Head loss


Flow Rate, Q

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at the beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## VIVA:

6. What is the relationship between friction head loss and pipe diameter?
7. What is the relationship between friction head loss and flow velocity?
8. What is the relationship between friction head loss and pipe length?
9. If flow velocity doubles, what happen to the frictional head loss?

## RESULT:

The mean value of minor losses of given set of pipes is obtained as, $\mathrm{f}=$

## TASK:

1. State Bernoulli's principle and Bernoulli's equation.
2. Make a model, indicating the flow of air through a venturi meter.
3. Justify the model by validating the model making.
4. Enumerate in the model about the narrow section during the flow through pipes.
5. Enumerate on the application of Bernoulli effect in IC engines, make a prototype / model and explain at least one application in IC engines.

## EX.NO:6

DATE :

## PRIOR LEARNING:

1. Pump
2. Centrifugal force
3. Discharge

OBJECTIVE:
To study the performance characteristics of a centrifugal pump and to determine the characteristic with maximum efficiency.

## THEORY:

A centrifugal pump converts rotational energy, often from a motor, to energy in a moving fluid. A portion of the energy goes into kinetic energy of the fluid. Fluid enters axially through eye of the casing, is caught up in the impeller blades, and is whirled tangentially and radially outward until it leaves through all circumferential parts of the impeller into the diffuser part of the casing. The fluid gains both velocity and pressure while passing through the impeller. The doughnutshaped diffuser, or scroll, section of the casing decelerates the flow and further increases the pressure. It is important to note that the water is not pushed radially outward by centrifugal force (non-existent force), but rather by inertia, the natural tendency of an object to continue in a straight line (tangent to the radius) when traveling around circle. This can be compared to the way a spincycle works in a washing machine.


Centrifugal Pump

## Write the physical description of the following:

1. Diffuser
2. Casing
3. Impeller blades

## APPARATUS REQUIRED:

1. Centrifugal Pump Set-up.
2. Stop Watch.
3. Meter Scale.
4. Collecting tank.

## FORMULAE:

## 1. Actual Discharge ( $Q_{a}$ ):

$\mathrm{Q}_{\mathrm{a}}=\mathrm{AH} / \mathrm{t} \mathrm{in} \mathrm{m}^{3} / \mathrm{s}$
Where,
A = Area of the tank in $\mathrm{m}^{2}$
$\mathrm{H}=\mathrm{H}=$ Height of water rise for the given time 't second' in m,
$t=$ Time taken for ' $H$ ' $m$ rise of water in $s$

## 2. Input Power ( $\mathbf{P}_{\mathbf{i}}$ ):

$\mathbf{P}_{\mathbf{i}}=\left(\mathbf{3 6 0 0} \mathbf{x} \mathbf{N}_{\mathbf{r}}\right) /\left(\mathbf{N}_{\mathbf{c}} \mathbf{x} \mathbf{T}\right)$ in kW
Where,
$\mathrm{N}_{\mathrm{r}}=$ Number of revolution of Energy meter
$\mathrm{N}_{\mathrm{c}}=$ Energy meter constant in rev/Kw hr
$\mathrm{T}=$ Time taken for $\mathrm{N}_{\mathrm{r}}$ revolution of energy meter in s

## 3. Output power ( $\mathbf{P}_{0}$ ) :

$\mathbf{P}_{\mathbf{0}}=\mathbf{w} \mathbf{Q}_{\mathrm{a}} \mathbf{h} / \mathbf{1 0 0 0}$ in kW
Where,
$\mathrm{w}=$ Specific weight of water in $\mathrm{N} / \mathrm{m}^{3}$
$\mathrm{Q}_{\mathrm{a}}=$ Actual discharge in $\mathrm{m}^{3} / \mathrm{s}$
$\mathrm{h}=$ Total head in m

## 4. Total Head (h):

$\mathrm{h}=\mathrm{h}_{\mathrm{d}}+\mathrm{h}_{\mathrm{s}}$ in m
Where,
$\mathrm{h}_{\mathrm{d}}=$ Delivery head in m
$\mathrm{h}_{\mathrm{S}}=$ Suction head in m

## 5. Efficiency ( $\eta$ ) : <br> $\boldsymbol{\eta}=\left(\mathbf{P}_{\mathbf{o}} / \mathbf{P}_{\mathbf{i}}\right) \times 100$ in $\%$



## CALCULATION

| S. No. | Expression | Formula/value | Unit |
| :---: | :--- | :--- | :--- |
| 1 | Suction head, $\mathrm{h}_{\mathrm{s}}$ |  |  |
| 2 | Delivery head, $\mathrm{h}_{\mathrm{d}}$ |  |  |
| 3 | Specific weight of water, w |  |  |
| 4 | Actual Discharge, $\mathrm{Q}_{\mathrm{a}}$ |  |  |
| 5 | Output power, $\mathrm{P}_{\mathrm{o}}$ |  |  |
| 6 | Energy meter constant, $\mathrm{N}_{\mathrm{c}}$ |  |  |$\quad$|  |
| :--- |
| 7 |
| Number of revolution of |
| Energy meter, $\mathrm{N}_{\mathrm{r}}$ |

WORKING PROCEDURE:

## GRAPH:

1. Discharge Vs Head
2. Discharge Vs Input
3. Discharge Vs Efficiency

## Viva

1.What is a pump?
2.What is a centrifugal pump?
3.what are forces involved in impeller?
4.What is priming?

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at the beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.


## RESULT:

Thus the performance characteristics of centrifugal pump was studied and the maximum efficiency was found to be $\qquad$ \%

## EX.NO:7

PERFORMANCE TEST ON RECIPROCATING PUMP
DATE :

## PRIOR LEARNING:

1. Energy meter
2. Pumps
3. Air vessels

## OBJECTIVE:

To conduct the performance test on reciprocating pump and to draw the characteristic curves.

## THEORY:

Reciprocating pumps are positive displacement pump as a definite volume of liquid is trapped in a chamber which is alternatively filled from the inlet and empited at a higher pressure through the discharge. The fluid enters a pumping chamber through an inlet and is pushed out through outlet valve by the action of piston. They are either single acting independent suction and delivery strokes or double acting suction and delivery both the directions. Reciprocating pumps are self priming pumps and are suitable for very high head at low flows. They deliver reliable discharge and it is often used for metering duties because of constancy of flow rate.


## Reciprocating Pump

Write the physical description of the following:

1. Suction Pipe:
2. Delivery Pipe:

## 3. Piston Cylinder Arrangement:

## APPARATUS REQUIRED:

1. Reciprocating Pump Set-up.
2. Stop Watch.
3. Meter Scale.
4. Collecting tank.
5. Collecting tank fitted with piezometer.
6. Differential U-tube mercury manometer.

## FORMULAE:

## 1. Actual Discharge $\left(\mathrm{O}_{\mathrm{a}}\right)$ :

$$
\mathrm{Q}_{\mathrm{a}}=\mathrm{AH} / \mathrm{tin} \mathrm{~m}^{3} / \mathrm{s}
$$

Where,
A = Area of the tank in $\mathrm{m}^{2}$
$\mathrm{H}=$ Height of water rise for the given time 't second' in m,
$t=$ Time taken for ' $H$ ' $m$ rise of water in $s$

## 2. Input Power ( $\mathbf{P}_{\mathrm{i}}$ ):

$\mathbf{P}_{\mathbf{i}}=\left(\mathbf{3 6 0 0} \times \mathbf{N}_{\mathbf{r}}\right) /\left(\mathbf{N}_{\mathbf{c}} \mathbf{x} \mathbf{T}\right)$ in kW
Where,
$\mathrm{N}_{\mathrm{r}}=$ Number of revolution of Energy meter.
$\mathrm{N}_{\mathrm{c}}=$ Energy meter constant rev/Kw hr.
$\mathrm{T}=$ Time taken for $\mathrm{N}_{\mathrm{r}}$ revolution of energy meter in s .

## 3. Output power ( $\mathbf{P}_{0}$ ):

$\mathbf{P}_{\mathbf{o}}=\mathbf{w} \mathbf{Q}_{\mathrm{a}} \mathbf{h} / \mathbf{1 0 0 0}$ in kW
Where
$\mathrm{w}=$ Specific weight of water in
$\mathrm{Q}_{\mathrm{a}}=$ Actual discharge in $\mathrm{m}^{3} / \mathrm{s}$
$\mathrm{h}=$ Total head in m

## 4. Total Head (h):

$\mathrm{h}=\mathrm{h}_{\mathrm{d}}+\mathrm{h}_{\mathrm{S}}$ in m
Where,
$\mathrm{h}_{\mathrm{d}}=$ Delivery head in m
$\mathrm{h}_{\mathrm{S}}=$ Suction head in m

## 5. Efficiency ( $\boldsymbol{\eta}$ ) :

$\boldsymbol{\eta}=\left(\mathbf{P}_{\mathbf{o}} / \mathbf{P}_{\mathbf{i}}\right) \mathbf{x} \mathbf{1 0 0}$ in $\%$

## WORK SPACE:



CALCULATION:

| S. No. | Expression | Formula/value | Unit |
| :---: | :---: | :---: | :---: |
| 1 | Suction head, $\mathrm{h}_{\mathrm{s}}$ |  |  |
| 2 | Delivery head, $\mathrm{h}_{\mathrm{d}}$ |  |  |
| 3 | Specific weight of water, w |  |  |
| 4 | Actual Discharge, $\mathrm{Q}_{\mathrm{a}}$ |  |  |
| 5 | Output power, $\mathrm{P}_{\text {o }}$ |  |  |
| 6 | Energy meter constant, $\mathrm{N}_{\mathrm{c}}$ |  |  |
| 7 | Number of revolution of Energy meter, $\mathrm{N}_{\mathrm{r}}$ |  |  |
| 8 | Input Power, $\mathrm{P}_{\mathrm{i}}$ |  |  |
| NOTE: |  |  |  |
| 1 mm of $\mathrm{Hg}=13.6 / 1000 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$ |  |  |  |

## WORKING PROCEDURE:

## GRAPH:

1. Discharge Vs Head
2. Discharge Vs Input
3. Discharge Vs Efficiency

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at te beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## VIVA:

1. What is the reason for internal cavitation in reciprocating pumps?
2. What is cavitation?
3. In what way the fluid displacement is engaged in a double acting reciprocating pump?
4. How cavitation can be eliminated in pumps?
5. What is suction head and acceleration head in pumps?

## RESULT:

The performance characteristic of the reciprocating pump is studied and the maximum efficiency is calculated $\qquad$ \%

## EX.NO:8 <br> PERFORMANCE TEST ON FRANCIS TURBINE <br> DATE :

PRIOR LEARNING:

1. Reaction turbine
2. Force Exerted by the jet on moving curved plate

## OBJECTIVE:

To conduct the performance test on Francis turbine at rated speed and at constant valve opening and to draw the characteristic curves.

## THEORY:

The Francis turbine is a reaction turbine, which means that the working fluid changes pressure as it moves through the turbine, giving up its energy. A casement is needed to contain the water flow. The turbine is located between the high pressure water source and the low pressure water exit, usually at the base of a dam. The inlet is spiral shaped. Guide vanes direct the water tangentially to the runner. This radial flow acts on the runner vanes, causing the runner to spin. The guide vanes (or wicket gate) may be adjustable to allow efficient turbine operation for a range of water flow conditions. At the exit, water acts on cup shaped runner features, leaving with no swirl and very little kinetic or potential energy. The turbine's exit tube is specially shaped to help decelerate the water flow and recover kinetic energy.


Francis Turbine
Write the physical description of the following:

1. Spiral Casing:
2. Guide Vanes:

## 3. Draft Tube:

## APPARATUS REQUIRED:

1. Francis turbine Set-up.
2. Tachometer
3. Dead weights
4. Stop watch

## FORMULAE:

## 1. Actual Discharge ( $\mathbf{Q}_{\mathrm{a}}$ ):

$$
\mathbf{Q}_{\mathrm{a}}=\mathbf{C}_{\mathrm{d}} \cdot \mathbf{a}_{1} \mathbf{a}_{2} \sqrt{ }(\mathbf{2 g h}) / \sqrt{ }\left(\mathbf{a}_{1}{ }^{2} \cdot \mathbf{a}_{2}{ }^{2}\right) \text { in } \mathrm{m}^{3} / \mathrm{s}
$$

where,
$\mathrm{a}_{1}=$ Area of the venturimeter inlet in $\mathrm{m}^{2}$
$\mathrm{a}_{2}=$ Area of the venturimeter throat in $\mathrm{m}^{2}$
$\mathrm{h}=$ Total head in m

## 2. Input Power ( $\mathbf{P}_{\mathrm{i}}$ ):

## $\mathbf{P}_{\mathbf{i}}=\mathbf{w} \mathbf{Q}_{\mathbf{a}} \mathbf{h}$ in watts

where,
$\mathrm{w}=$ Specific weight of water
$\mathrm{Q}_{\mathrm{a}}=$ Actual discharge in $\mathrm{m}^{3} / \mathrm{s}$
$\mathrm{h}=$ Total head in m

## 3. Output power $\left(\mathbf{P}_{\mathbf{0}}\right)$ :

$P_{0}=(2 x \pi N T) / 60$ in watts where,
$\mathrm{N}=$ speed in r.p.m
$\mathrm{T}=$ Torque in $\mathrm{N}-\mathrm{m}$

## 4. Torque ( T) :

$$
\mathbf{T}=[(\mathbf{W}-\mathbf{S})(\mathbf{D}+\mathbf{d}) .9 .81] / 2 \text { in } \mathrm{N}-\mathrm{m}
$$

where,
$\mathrm{W}=$ Dead weight in hanger in kg
$\mathrm{S}=$ Spring balance reading in kg
$\mathrm{D}=$ Diameter of brake drum in m
$d=$ Diameter of rope in $m$
5. Efficiency ( $\boldsymbol{\eta}$ ) :

$$
\eta=\left(\mathbf{P}_{0} / \mathbf{P}_{i}\right) \times 100 \%
$$

WORK SPACE:


CALCULATION:

| S. No. | Expression | Formula/value | Unit |
| :---: | :---: | :---: | :---: |
| 1 | Diameter of inlet, $\mathrm{d}_{1}$ |  |  |
| 2 | Area of inlet, $\mathrm{a}_{1}$ |  |  |
| 3 | Diameter of throat, $\mathrm{d}_{2}$ |  |  |
| 4 | Area of throat, $\mathrm{a}_{2}$ |  |  |
| 5 | Coefficient of discharge, $\mathrm{C}_{\mathrm{d}}$ |  |  |
| 6 | Difference in manometer level, x |  |  |
| 7 | Differential head, h |  |  |
| 8 | Actual Discharge, $\mathbf{Q}_{\mathbf{a}}$ |  |  |
| 9 | Input Power, $\mathrm{P}_{\mathrm{i}}$ |  |  |
| 10 | Torque ( T ) |  |  |
| 11 | Output power ( $\mathrm{P}_{\mathrm{o}}$ ) |  |  |
| 12 | Efficiency ( $\eta$ ) |  |  |

NOTE:
1 mm of $\mathrm{Hg}=13.6 / 1000 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$
$1 \mathrm{~kg} / \mathrm{cm}^{2}=10.33 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$

## WORKING PROCEDURE:

## GRAPH:

1. Discharge Vs Total Head
2. Discharge Vs Output Power
3. Discharge Vs Efficiency

## VIVA:

I. What is radial flow turbine?
II. What is the difference between impulse and reaction turbine?
III. What is Hydraulic Efficiency?
IV. What is Mechanical Efficiency?
V. What is Volumetric Efficiency?
VI. What is Overall Efficiency?

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at te beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## RESULT:

Thus the performance characteristics of the Francis wheel turbine are done and the maximum efficiency of the turbine is $\qquad$ \%

## PRIOR LEARNING:

1. Turbine
2. Force Exerted by the jet on moving curved plate

## OBJECTIVE:

To conduct the performance test on Pelton wheel turbine at rated speed and at constant spear opening and to draw the characteristic curves.

## THEORY:

Pelton Turbine is a Tangential flow impulse turbine in which the pressure energy of water is converted into kinetic energy to form high speed water jet and this jet strikes the wheel tangentially to make it rotate. It is also called as Pelton Wheel.


Pelton Wheel

Write the physical description of the following:

1. Nozzle:
2. Casing:

## 3. Breaking jet:

## APPARATUS REQUIRED:

1. Pelton wheel Set-up.
2. Tachometer
3. Dead weights
4. Stop watch.

## FORMULAE:

1. Actual Discharge $\left(\mathrm{Q}_{\text {act }}\right)$ :
$Q_{\text {act }}=C d . a_{1} a_{2} \sqrt{ }(2 \mathrm{gh}) / \sqrt{ }\left(\mathrm{a}_{1}{ }^{2}-\mathrm{a}_{2}{ }^{2}\right)$ in $\mathrm{m}^{3} / \mathrm{s}$ where,
$a_{1}=$ Area of the venturimeter inlet in $\mathrm{m}^{2}$
$\mathrm{a}_{2}=$ Area of the venturimeter throat in $\mathrm{m}^{2}$
$\mathrm{h}=$ Total head in m
2. Input Power ( Pi ):

$$
\mathrm{P}_{\mathrm{i}}=\mathrm{w} \mathrm{Q}_{\text {act }} \mathrm{H} / 1000 \text { in } \mathrm{kW}
$$

where,
$\mathrm{w}=$ Specific weight of water
$\mathrm{Q}_{\text {act }}=$ Actual discharge in $\mathrm{m}^{3} / \mathrm{s}$
$\mathrm{H}=$ Total head in m of water
3. Output power ( Po ) :
$\mathrm{P}_{\mathrm{o}}=(2 \pi \mathrm{NT}) / 60000$ in kW
Where,
$\mathrm{N}=$ speed in rpm
$\mathrm{T}=$ Torque in $\mathrm{N}-\mathrm{m}$
4. Torque ( T ) :

$$
\mathrm{T}=[(\mathrm{M}-\mathrm{S})(\mathrm{D}+\mathrm{d}) \mathrm{x} 9.81] / 2 \text { in } \mathrm{N}-\mathrm{m}
$$

where,
$\mathrm{M}=$ Mass of hanger in kg
$\mathrm{S}=$ Spring balance reading in kg
$D=$ Diameter of brake drum in $m$
$d=$ Diameter of rope in $m$
5. $\quad$ Efficiency $(\eta)$ :

$$
\eta=\left(P_{o} / P_{i}\right) \times 100 \%
$$



CALCULATION

| S. No. | Expression | Formula/value | Unit |
| :---: | :---: | :---: | :---: |
| 1 | Diameter of inlet, $\mathrm{d}_{1}$ |  |  |
| 2 | Area of inlet, $a_{1}$ |  |  |
| 3 | Diameter of throat, $\mathrm{d}_{2}$ |  |  |
| 4 | Area of throat, $\mathrm{a}_{2}$ |  |  |
| 5 | Coefficient of discharge, $\mathrm{C}_{\mathrm{d}}$ |  |  |
| 6 | Differential head, h |  |  |
| 7 | Actual Discharge, $\mathbf{Q a}_{\mathbf{a}}$ |  |  |
| 8 | Input Power, $\mathrm{P}_{\mathrm{i}}$ |  |  |
| 9 | Torque ( T ) |  |  |
| 10 | Output power ( $\mathrm{P}_{\mathrm{o}}$ ) |  |  |
| 11 | Efficiency ( $\eta$ ) |  |  |

NOTE:
1 mm of $\mathrm{Hg}=13.6 / 1000 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$
$1 \mathrm{~kg} / \mathrm{cm}^{2}=10.33 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$

## WORKING PROCEDURE:

## GRAPH:

1. Load Vs Total Head
2. Load Vs Output Power
3. Load Vs Efficiency

VIVA:

1. What are the Advantages of Pelton wheel?
2. What are the Disadvantages of Pelton wheel?
3. What are the application of Pelton wheel Application?
4. Where do you prefer Pelton Wheel?
5. What are the main components of Pelton wheel?

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at te beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## RESULT:

Thus the performance characteristics of the Pelton Wheel Turbine are done and the maximum efficiency of the turbine is $\qquad$ \%

## EX.NO:10 <br> PERFORMANCE TEST ON KAPLAN TURBINE <br> DATE :

## PRIOR LEARNING:

1. Axial flow reaction turbine
2. Force Exerted by the jet on moving curved plate

## OBJECTIVE:

To conduct the performance test on Kaplan turbine at rated speed and at constant spear opening and to draw the characteristic curves.

## THEORY:

Kaplan turbine is a propeller type, axial flow water turbine. It has adjustable blades. Adjustable blades help in obtaining good efficiency at wide range of head and discharge available to the turbine. It is suitable for low head and high flow applications. It is an evolution of Francis turbine. It is designed to operate where head is low. It has automatically adjusted propeller blades with automatically adjusted wicket gates to achieve efficiency over a wide range of flow and water level. The Kaplan turbine was an evolution of the Francis turbine. Its invention allowed efficient power production in low-head applications which was not possible with Francis turbines. The head ranges from 10-70 metres and the output ranges from 5 to 200 MW . Runner diameters are between 2 and 11 metres. Turbines rotate at a constant rate, which varies from facility to facility. That rate ranges from as low as 54.5 rpm to 450 rpm .


## Kaplan Turbine

Write the physical description of the following:

1. Scroll casing:
2. Hub with vanes or runner of the turbine:
3. Draft tube:

## APPARATUS REQUIRED:

1. Kaplan turbine Set-up.
2. Tachometer
3. Dead weights
4. Stop watch

## WORKING PROCEDURE:

WORK SPACE:

## FORMULAE:

## 1. Actual Discharge $\left(Q_{a}\right)$ :

$\mathbf{Q}_{\mathrm{a}}=\mathbf{C}_{\mathrm{d}} \cdot \mathbf{a}_{1} \mathbf{a}_{2} \sqrt{ }(\mathbf{2 g h}) / \sqrt{ }\left(\mathbf{a}_{1}{ }^{2}-\mathbf{a}^{2}{ }^{2}\right) \mathrm{in}^{3} / \mathrm{s}$ where,
$\mathrm{a}_{1}=$ Area of the venturimeter inlet in $\mathrm{m}^{2}$
$\mathrm{a}_{2}=$ Area of the venturimeter throat in $\mathrm{m}^{2}$
$\mathrm{h}=$ Total head in m

## 2. Input Power ( $\mathbf{P}_{\mathbf{i}}$ ):

$\mathbf{P i}=w$ Qah in watts where,
$\mathrm{w}=$ Specific weight of water
$\mathrm{Q}_{\mathrm{a}}=$ Actual discharge in $\mathrm{m}^{3} / \mathrm{s}$
$\mathrm{h}=$ Total head in m

## 3. Output power ( $\mathbf{P}_{0}$ ) :

$P_{0}=(2 x \pi N T) / 60$ in watts where,
$\mathrm{N}=$ speed in r.p.m
$\mathrm{T}=$ Torque in $\mathrm{N}-\mathrm{m}$

## 4. Torque ( T) :

$\mathrm{T}=[(\mathrm{W}-\mathrm{S})(\mathrm{D}+\mathrm{d}) .9 .81] / 2$ in $\mathrm{N}-\mathrm{m}$ where,
$\mathrm{W}=$ Mass of hanger in kg
$\mathrm{S}=$ Spring balance reading in kg
$\mathrm{D}=$ Diameter of brake drum in m $d=$ Diameter of rope in $m$
5. Efficiency ( $\boldsymbol{\eta}$ ) :
$\eta=\left(\mathrm{P}_{\mathrm{o}} / \mathrm{P}_{\mathrm{i}}\right) \times 100$ in $\%$

CALCULATION

| S. No. | Expression | Formula/value | Unit |
| :---: | :---: | :---: | :---: |
| 1 | Diameter of inlet, $\mathrm{d}_{1}$ |  |  |
| 2 | Area of inlet, $a_{1}$ |  |  |
| 3 | Diameter of throat, $\mathrm{d}_{2}$ |  |  |
| 4 | Area of throat, $\mathrm{a}_{2}$ |  |  |
| 5 | Coefficient of discharge, $\mathrm{C}_{\mathrm{d}}$ |  |  |
| 6 | Difference in manometer level, x |  |  |
| 7 | Differential head, h |  |  |
| 8 | Actual Discharge, $\mathbf{Q}_{\mathbf{a}}$ |  |  |
| 9 | Input Power, $\mathrm{P}_{\mathrm{i}}$ |  |  |
| 10 | Output power ( $\mathrm{P}_{\mathrm{o}}$ ) |  |  |
| 11 | Torque ( T ) |  |  |
| 12 | Efficiency ( $\eta$ ) |  |  |

## NOTE:

1 mm of $\mathrm{Hg}=13.6 / 1000 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$
$1 \mathrm{~kg} / \mathrm{cm}^{2}=10.33 \mathrm{~m}$ of $\mathrm{H}_{2} \mathrm{O}$

## GRAPH:

1. Discharge Vs Total Head
2. Discharge Vs Output Power
3. Discharge Vs Efficiency

VIVA:

1. What is function of draft tube?
2. Why draft tube is used in Kaplan turbine?
3. How are hydraulic turbine classified?
4. What is Mechanical Efficiency?
5. What is Volumetric Efficiency?
6. What is Overall Efficiency?

## SAFETY INSTRUCTIONS:

- Make sure that all the valves are closed at te beginning of the experiment.
- The discharge valve and collecting tank valve should be closed when starting the experiment.
- Care must be taken to ensure that the collecting tank valve is opened at appropriate time to ensure that water doesn't overflow.
- Leaning and looking into the collecting tank should be avoided.


## RESULT:

Thus the performance characteristics of the Kaplan turbine are done and the maximum efficiency of the turbine is $\qquad$ \%

## TASK:

1. Enumerate the operation theory of impulse and reaction turbine
2. Make a model on Pelton turbine
3. Make a model on wind turbine
4. Discuss on Helicopter shaft rotation (coaxial contra rotation) and compare with the turbine wheel rotation
5. Make a comparison table on at least three rotary machines application, principle and power produced.
