

## Thermodynamic Properties of Steam :-

Various properties  $\rightarrow$  Various condition of steam (wet, dry & superheated)

### i) Enthalpy of steam (h) :-

It is the amount of heat added to the water from freezing point to till the water.

$$\text{Wet steam, } h_w = h_f + x h_{fg}.$$

$$\text{Dry steam, } h_d = h_f + h_{fg} = h_g.$$

$$\text{Superheated steam, } h_{sup} = h_g + C_{ps} (T_{sup} - T_{sat})$$

where,

$$h = \text{Enthalpy of steam KJ/kg.}$$

$$T_{sup} - T_{sat} = \text{degree of superheat.}$$

$$C_{ps} = \text{Specific heat of superheated steam } (2 - 2.3 \text{ KJ/kg.K})$$

### ii) Specific Volume of Steam (v).

It is defined as the volume occupied by the unit mass of steam at the given pressure and temperature.

$$\text{Wet steam } v_w = x v_g.$$

$$\text{Dry steam } v_d = v_g.$$

$$\text{Superheated steam } v_{sup} = \frac{v_g T_{sup}}{T_{sat}}$$

$$v = \text{Specific volume (m}^3/\text{kg)}$$

$$w = \text{wet d - dry.}$$

$$\text{sup - superheated.}$$

$$h = \text{enthalpy.}$$

$$s = \text{entropy.}$$

### iii) Work done during expansions (W).

During the evaporation process, there is a considerable increase in its volume when the pressure remains constant.

Energy required for absorption of latent heat for increasing volume of the steam.

$$\text{Wet steam, } W_{wet} = 100 p x v_g.$$

$$\text{Dry steam } W_d = 100 p v_g \text{ KJ}$$

$$\text{Superheated steam } W_{sup} = 100 p v_{sup}.$$

where,

$$W = \text{work done in KJ.}$$

$$p = \text{pressure at which evaporation (bars)}$$

## iv) Internal Energy of steam (U) :

Internal energy of steam is defined as the actual heat energy stored in the steam above the freezing point of water at the given conditions.

$$h = W + \Delta u.$$

$$\Delta u = Wh - W$$

$$\text{Wet steam } U_w = [h_f + x h_{fg}] - [100 P x v_g].$$

$$\text{Dry steam } U_d = [h_f + h_g] - [100 \cdot P v_g]$$

$$\text{Superheated steam } U_{sup} = h_{sup} - 100 P v_g$$

## v) Entropy of steam (S).

It is the property of the steam which increases with increase in temperature and decrease with decrease in temperature.

$$\text{Wet steam } S_{wet} = S_f + x S_g.$$

$$\text{Dry steam } S_{dry} = S_f + S_g.$$

$$\text{Superheated steam } S_{sup} = S_g + C_{ps} \left( \frac{T_{sup}}{T_{sat.}} \right).$$

## Determination of Dryness fraction :-

It is defined as the ratio of the mass of actual dry steam to the mass of same quantity of wet steam.

$$X = \frac{m_g}{m_g + m_f}.$$

where

$m_g$  = Mass of actual dry steam.

$m_f$  = Mass of water in suspension

$m$  = mass of wet steam. =  $m_g + m_f$ .

## Latent heat of Evaporation:-

The amount of heat added during heating of water from boiling point to dry saturated state is called latent heat of evaporation (or) enthalpy of vaporization.

9. A rigid tank  $1 \text{ m}^3$  contain liquid vapour mixture of water at  $1.5 \text{ MPa}$  and dryness fraction of  $0.9$ . The mixture of is allowed to cool until the pressure is reduced to  $300 \text{ kPa}$ . Determine the final mass of liquid, mass of vapour and heat transfer during the process.

Given :-

$$V = 1 \text{ m}^3$$

$$P = 1.5 \text{ MPa} = 15 \text{ bar}$$

$$x = 0.90$$

To find :-

$m_f$ ,  $m_g$  &  $Q$

$$\therefore m = m_f + m_g$$

$$m = x m_g$$

Solution :-

Initially the steam is wet condition.

$$v_f = 0.001154 \text{ m}^3/\text{kg}$$

$$v_g = 0.13167 \text{ m}^3/\text{kg}$$

$$h_f = 844.6 \text{ kJ/kg}$$

$$h_{fg} = 1945.3 \text{ kJ/kg}$$

$$h_{wet} = h_f + x h_{fg}$$

$$= 844.6 + 0.90 (1945.3)$$

$$= 2595.37 \text{ kJ/kg}$$

$$v_{wet} = x v_g$$

$$= 0.9 (0.13167)$$

$$= 0.1185 \text{ m}^3/\text{kg}$$

At  $300 \text{ kPa}$  ( $3 \text{ bar}$ )

$$v_f = 0.001074 \text{ m}^3/\text{kg}$$

$$v_g = 0.60553 \text{ m}^3/\text{kg}$$

$$h_f = 561.5 \text{ kJ/kg}$$

$$h_{fg} = 2163.2 \text{ kJ/kg}$$

It is rigid tank, its volume does not change.

$$v_1 = v_2 = v_g \cdot x$$

$$0.1185 = 0.60553 \times x$$

$$x = 0.1957$$

$$h_2 = h_{f_2} + x_2 h_{fg_2}$$

$$= 561.5 + 0.1957 (2163.2)$$

$$h_2 = 984.83 \text{ kJ/kg}$$

$$m = \frac{V}{v_{wet}} = \frac{1}{0.1187} = 8.4245 \text{ kg.}$$

Mass of Vapour

$$m_g = x \cdot m \quad = \text{formula check.}$$

$$= 0.1957 + 8.4245 \quad = \underline{\underline{9}}$$

$$m_g = 1.648 \text{ kg.}$$

mass of liquid.

$$m = m_f + m_g.$$

$$m_f = m - m_g.$$

$$= 8.4245 - 1.648.$$

$$m_f = 6.775 \text{ kg.}$$

Total Heat transfer.

$$Q = m (h_2 - h_1)$$

$$= 1 (984.83 - 2595.37)$$

$$Q = -1610.54 \text{ kJ/kg.}$$

This negative sign indicates that heat is transferred from the tank to surrounding.

10) Steam initially at 0.3 MPa, 250°C is cooled at constant volume.

a) At what temperature will the steam become saturated vapour.

b) What is the quality of steam at 80°C?

c) What is the heat transferred per kg of steam in cooling from 250°C to 80°C?

Given :-

$$P_1 = 0.3 \text{ MPa} = 3 \text{ bar}$$

$$T = 250^\circ\text{C}$$

$$T_2 = 80^\circ\text{C}$$

Solution :-

a) At 3 bar, from steam table.

$$T_{sat} = 133.5^\circ\text{C}$$

$$T = 250^\circ\text{C}$$

$$T > T_{sat}$$

The condition of the steam is superheated.

At 3 bar, 25°C in enthalpy superheated steam.

(13)

$$h_1 = 2967.6 \text{ kJ/kg}$$

$$v_1 = 0.7964 \text{ m}^3/\text{kg}$$

At constant volume,

$$v_1 = v_2 = 0.7964 \text{ m}^3/\text{kg}$$

When  $v_g = 0.7964$ ,  $t_{\text{sat}} = 123.9^\circ\text{C}$ .

steam became saturated vapour at  $t = 123.9^\circ\text{C}$ .

b) At 80°C, quality of steam.

$$v_f = 0.001029 \text{ m}^3/\text{kg} \quad h_f = 334.91 \text{ kJ/kg} \quad P_2 = 0.4736 \text{ bar}$$

$$v_g = 3.4091 \text{ m}^3/\text{kg} \quad h_{fg} = 2308.8 \text{ kJ/kg}$$

$$\begin{aligned} v_1 = v_2 = x v_g & \quad \left| \quad v_1 = v_2 = v_f + x \cdot v_g \right. \\ 0.7964 = x \cdot 3.4091 & \quad \left| \quad = 0.001029 + x(3.4091 - 0.001029) \right. \\ x = 0.2336 & \quad \left| \quad x = 0.234 \right. \end{aligned}$$

$$\begin{aligned} h_2 &= h_f + x_2 h_{fg} \\ &= 334.91 + (0.234)(2308.8) \end{aligned}$$

$$h_2 = 874.28 \text{ kJ/kg}$$

Heat Transfer,

$$Q = u_2 - u_1 \quad \Rightarrow \quad h = u + Pv$$

$$Q = (h_2 - P_2 v_2) - (h_1 - P_1 v_1) \quad u = h - Pv$$

$$= h_2 - P_2 v_2 - h_1 + P_1 v_1$$

$$Q = \text{[scribble]}$$

$$Q = (h_2 - h_1) + v(P_1 - P_2)$$

$$Q = (874.28 - 2967.6) + 0.7964(3 - 0.4739)$$

$$Q = -2091.7 + 2011.5$$

$$Q = -2089.682 \text{ kJ/kg}$$

$$Q = -2089.6 \text{ kJ/kg}$$

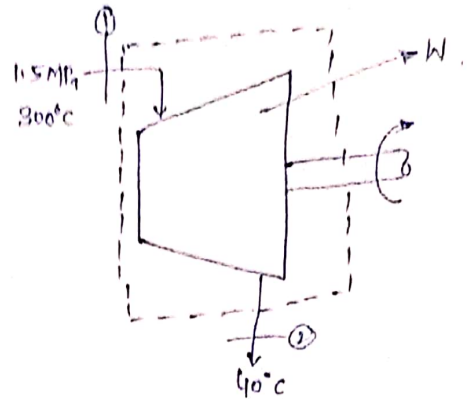
(11) Steam initially at 1.5 MPa, 300°C expands reversibly and adiabatically in a steam turbine to 40°C. Determine the ideal work output of the turbine per kg of steam.

Given :-

$$P = 1.5 \text{ MPa} = 15 \text{ bar}$$

$$T_1 = 300^\circ\text{C}$$

$$T_2 = 40^\circ\text{C}$$



To find :-

Work output of the turbine.

Solution :-

Formula, @ turbine. in SFEE.

$$W = h_1 - h_2$$

$$T = 300^\circ\text{C}, T_{\text{sat}} = 198.3$$

$$T > T_{\text{sat}}$$

Steam condition is superheated

$$\text{At } 40^\circ\text{C}, P_{\text{sat}} = 7.375 \text{ kPa}, S_f = 0.572 \text{ kJ/kg}\cdot\text{K}$$

$$= 0.07375 \text{ bar}, h_f = 167.5 \text{ kJ/kg}$$

$$V_g = 19.546 \text{ m}^3/\text{kg}, h_{fg} = 2406.9 \text{ kJ/kg}$$

$$S_{fg} = 7.686 \text{ kJ/kg}\cdot\text{K}$$

At  $t = 300^\circ\text{C}$

$$S_1 = 6.921 \text{ kJ/kg}\cdot\text{K}, h_1 = 3038.9 \text{ kJ/kg}$$

$$S_1 = S_2$$

$$S_1 = S_f + x_2 \cdot S_{fg}$$

$$6.921 = 0.572 + x_2 (7.686)$$

$$x_2 = 0.826$$

$$h_2 = h_f + x_2 h_{fg}$$

$$= 167.5 + 0.826 (2406.9)$$

$$h_2 = 2155.5 \text{ kJ/kg}$$

$$W = h_1 - h_2 = 3038.9 - 2155.5$$

$$W = 883.4 \text{ kJ/kg}$$