



Fluid Mechanics and Machinery— **UNIT IV PUMPS**

Topic-Impact of jets – Theory of roto-dynamic machines-Various efficiencies

The hydraulic machine which Convert the mechanical energy into hydrenlic energy are Called pumps.

Mechanical Energy > Hydraulic Energy.

Hydraulic energy is in the form of Pressure energy.

major types: 1. Centrifugal pump 2. Reciprocating pump. when the mechanical energy is Converted into Pressure energy by means of Centrifugal force acting on the flid, the hydrautic m/c is called centrifugal brunk.

Mechanical Energy is Converted into hydranlie energy or Pressure energy by Sucking the ligis of into a cylinder in which a piston is reinprocating (moving backwards and forwards, which exects the thrust on the liquid and increases its hydraule energy (Pressure onergy)



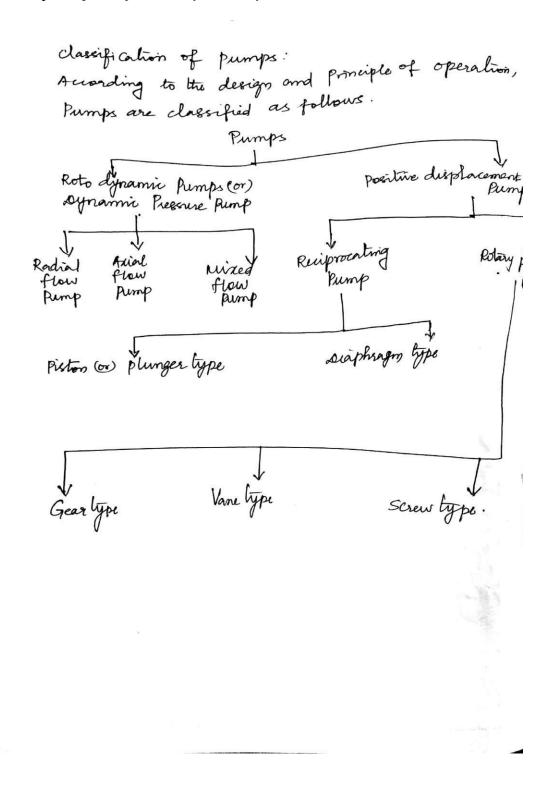
SNSCOLLEGEOFTECHNOLOGY, COIMBATORE-35





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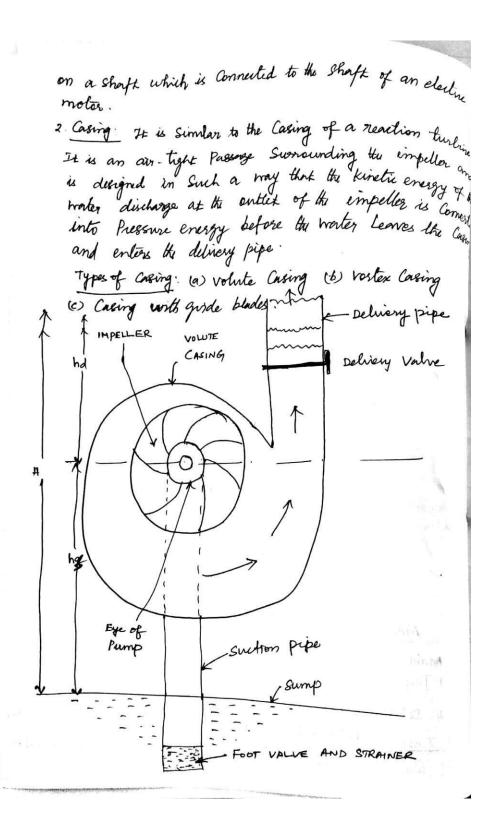
The Centrifugal pumps acts as a neverse of an invased radial flow reaction turbine. The flow in Centrifugal pumps is in the radial - The Centrifugal pump works on the principles of forced vortex flew which means that when a certain mass of ligited is rolated by an external torque, the rise in Pressure head of the rotating liquid takes place. I The rise in Pressure head at any point of the rotating liquid is propostional to the Square of langential velocity of the ligisod at that point Rise in Pressure head = $\frac{V^2}{2g}$ (or) $\frac{W^2r^2}{2g}$ I Thus at the outlet of the impeller, where radius is more: the rise in pressure head will be more and the liquid will be discharged at the outlet with a high pressure head. - sue to this high Pressure head, the liquid Com be lifted to high level. MAIN PARTS OF A CENTRIFUGAL PUMP Main Parts: 1. Impeller 2. Casing 3. Suction pipe with a foot value 4. Delivery pipe. 1. Impellor: Rotating Part of a Centrifugal pump is Called impelle It Consuls of a backward Curred Vanes. The impeller is mounted





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CACING: Volute Casing, which Surrounds the impeller. It is of Spiral type in which area of flow increases gradually. The increase in area of flow decreases to velocity of flow. The decrease in velocity increases the Pressure of the water flowing through the Casing It has been observed that in Case of volute Casing, the afficiency of the pump increases Shightly as a large amount of energy is lost due to the formation of eddies in this type of Casing (b) vortex Caing: If a circular chamber is introduced between the casing and the impeller as the caving is known as voster casing the not efficiency when only Volute Coving is pravided. (c) Cosing with Gorde Blodes: The impeller is Surrounded by a series of gride blades mounted on a ring which is known as diffuser. (3) Suction pipe with a foot valve and a strainer: A pipe whose one end is connected to the inlet of the Rump and other and dips into mater in a Sump is Known as suction pipe. A foot value which is a non-return value er one-way type of value is fitted at the lower end of the Suction pipe. The foot value opens only in the upward direction A strainer is also fit at the lower end of the Suction pipe. (4) Delivery pipe: A pipe whose one end is Connected to the outlet of the pump and other end delivers the moter at a required height is known as delivery pipe





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DEFINITIONS OF HEAD AND EFFICIENCIES OF A CENTRIFUGAL
Suction Head (hs): It is the Vertical height of the Centre
line of the Centrifugal pump above the water Surface in the tank or pump from while water is to be lifted
This is also could as suchan life. between the Centre
line of the pump and the rection is known as delinery
(3) Shatic Head (Hs): The Sum of Suction head and delivery head is known as static head. This is represented by
(1) Manamater, Head (Am): The manometric head is defined
as the head against which a centrifugal pump has to work. It is denoted by (Hm). Expressed as
(a) Am = Head imparted by the impeller to the water -
Hm = $\frac{Vw_2 u_2}{g}$ _ Loss of head in impeller and Hm = $\frac{Vw_2 u_2}{g}$ _ Casing
(b) I'm = Total head at cuttet of the pump -
Total head at the inlet of the pump. $= \left(\frac{P_0}{f_0g} + \frac{V_0^2}{2g} + Z_0\right) - \left(\frac{P_i}{f_0g} + \frac{V_i^2}{2g} + Z_i\right) - 6$
where $\frac{P_0}{P_g}$ = Pressure head at entlet of the pump had
Vo2 = valority head at outlet of the pump
\frac{V_0^2}{2g} = velocity head at outlet of the pump = velocity head in delinery pipe \frac{Vd^2}{2g}

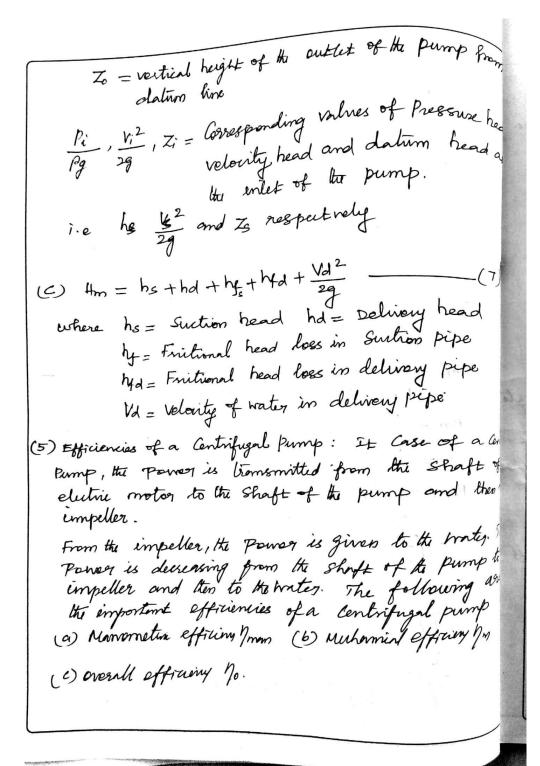




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(a) I main = Mannometric heard / Heard impasted by impellar to hoste
$$= \frac{Hm}{\sqrt{W_2 U_2}} = \frac{g Hm}{V_{W_2} U_2}$$
(8)

(c)
$$\eta_0 = \frac{h \text{ wight of mates Lifted} \times \frac{hm}{1000} = \frac{h \text{ Hm}}{1000}$$

 $\eta_0 = \eta_{\text{man}} \times \eta_{\text{m}}$.